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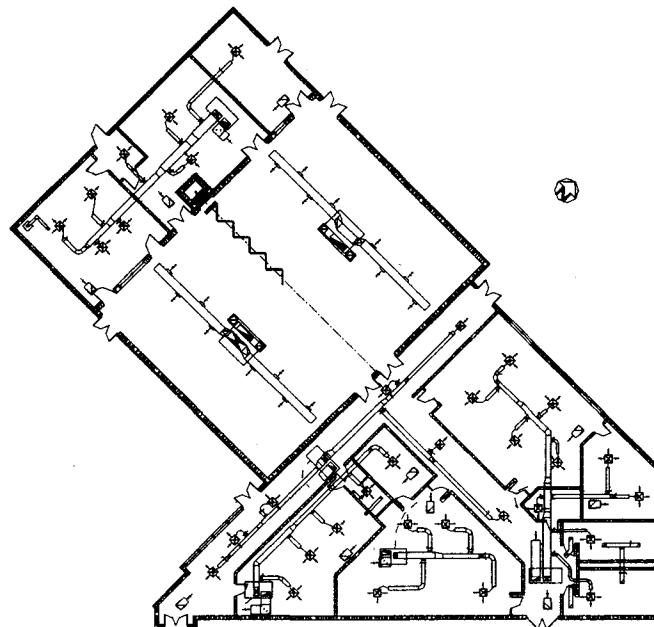
Advanced HVAC Design

Final Project



Sunset Elementary School

Addition



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12-10-01

Introduction:.....	2
Cooling Load Calculations:	3
Assumptions:.....	3
Heat Generations:.....	3
Heat Generation by Occupants:	3
Heat Addition from Appliances:.....	4
Infiltration:	5
Thermal Conduction Through the Walls and Roofs:	6
Heat Transfer Due to the Sun's Radiation:	10
Detailed Information of the Heat Generated for Different Times	11
Summary of the Cooling Load.....	17
Air Circulation and the Selection of the Unit:	18
Fresh Air Requirement:.....	18
Ratio of the fresh air	19
Selection of the Air Conditioning Unit.....	20
Estimate of Cost of the HVAC system:	21

Attachments:

Cut Sheets:

- Heavy Duty Gym Grills
- Supply Diffusers
- Return Grills
- Exhaust Fans
- Roof Top Unit Specifications

Figure 1: Floor Plan and Wall Types

Figure 2: Zone Layout

Figure 3: Roof Top Units

Figure 4: Tentative Schematic of Ducts

Figure 5: Ceiling Grid and Diffuser Layout

Figure 6: Details of Unit

Introduction:

Location:

Sunset Elementary School located on 1510 w. Republic Avenue in Salina Kansas.

Purpose:

We are going to completely design an HVAC system to satisfy the comfort conditions on September 1, 2001.

Design:

Our project is a 15,000 square foot expansion to the Sunset Elementary School. The floor plan (Fig.1) gives a complete layout of our project and shows which way the building is facing. The way the building is facing has a major impact on the design of the system. The walls were all measured and calculations were made for areas for the amount of exposure for each room.

The next phase of the project was to figure out the number of people and appliances that would be occupying the rooms during the specific day that we chose. All of these numbers can be found in References attachment and those numbers were used for the cooling loads calculations, which can be found in Cooling Load Calculations, which is attached. We then looked at the floor plan (Fig 1) and decided on the zones that we would use for the project. We decided the zones based on similarities in sun exposure and what the rooms would be used for. The zone plan drawing can be found in Fig. 2.

We also decided to go ahead and place the air conditioning units on our structure. The Trane units that we used (the exact unit number and size will be finalized for the final report) were place on the structure based on the structural integrity of the building and would be certified by a structural engineer before the design is completed. This drawing can be found in Fig. 3. We also drafted a tentative schematic of the ductwork that will used to deliver the conditioned air to our rooms and they can found on the Schematic Ductwork drawing in Fig. 4.

Figure 5 presents the ceiling grid and diffuser layout , and figure 6 provides the detailed of the units.

Cooling Load Calculations:

Before selecting the air conditioning unit for each zone, required capability of the units must be determined. The unit needs to be able to handle the cooling load for the zone, and also have the ability to provide enough air for the circulation, including adding fresh air from the outside while maintaining the temperature and moisture at a desired range. The cooling loads will be determined by considering these factors; occupants' activities, appliances being used, and heat from outside by conduction, radiation or infiltration.

Assumptions:

To calculate the cooling load, these assumptions are set.

- The gender ratio of the elementary school is 1:1. The values of heat generation from adult are the average of both male and female.
- Students are considered as children.
- The sample climate data of Salina, Kansas is based on the information provided by the National Weather Service of Wichita on September 1st, 2001. The enthalpy values include the components of water vapor.
 - 8:00 A.M.: $66^{\circ}\text{F}_{\text{DB}}$ $56^{\circ}\text{F}_{\text{WB}}$ $\Phi = 69\%$ $i = 23.6 \text{ Btu/lbm}_{\text{air}}$
 - 12:00 P.M.: $80^{\circ}\text{F}_{\text{DB}}$ $55^{\circ}\text{F}_{\text{WB}}$ $\Phi = 42\%$ $i = 23.2 \text{ Btu/lbm}_{\text{air}}$
 - 5:00 P.M.: $85^{\circ}\text{F}_{\text{DB}}$ $54^{\circ}\text{F}_{\text{WB}}$ $\Phi = 34\%$ $i = 23.0 \text{ Btu/lbm}_{\text{air}}$
- The climate condition of the building should be shown as below,
 - Target $73^{\circ}\text{F}_{\text{DB}}$ $58^{\circ}\text{F}_{\text{WB}}$ $\Phi = 40\%$ $i = 25.0 \text{ Btu/lbm}_{\text{air}}$
- The level of the activities in selected hours remains constant.

Heat Generations:

Heat Generation by Occupants:

The values of heat generated by people in the building for various activities are referred from the *Standard Handbook of Architectural Engineering*; table 4-11: Metabolism of Human Activities (P.480). The ratios of the sensible and latent heat for each activity are derived from the *ASHRAE handbook 2001*; table 8.18 (P.842). The table describes the values used for the evaluation.

Table A-1: Heat Generation in Sensible and Latent Heat

Activity:	Heat Generation (Btu/hr)			% Heat Ratio		Adult (Btu/hr)		Child (Btu/hr)	
	Male	Female	Child	Sensible	Latent	Sensible	Latent	Sensible	Latent
Walking	800	680	600	50%	50%	370	370	300	300
Eating	520	450	400	55%	45%	267	218	220	180
Cooking	1600	1300	N/A	40%	60%	580	870	N/A	N/A
Office work	640	540	N/A	55%	45%	325	266	N/A	N/A
Classroom	480	410	360	60%	40%	267	178	216	144

Heat Addition from Appliances:

Appliances being used during the hours are another factors adding heat into the rooms. All rooms have lighting, and in addition, each room has different kind of appliances. For example, the kitchen has refrigerator, freezer, dishwasher, hot / cold bars and ranges. The values are taken from the *ASHRAE handbook 2001*.

Table A-2: Table of Heat Generation from Appliances

Item	Location(s)	Heat (Btu/hr)
Large Refrigerator	Kitchen	300
Large Freezer	Kitchen	1,840
Dish Washer (Conveyer, Water Sanitized, Hood Equipped)	Kitchen	190
Range (Hot Box)	Kitchen	2,690
Food Warmer (Hot Bar)	Kitchen	990
Cold Bar (50% of hot bar)	Kitchen	495
Computer	Kitchen	1,366
Vending Machine	Staff Planning	940
Hand Dryer	Restrooms	1,366
Beverage Dispenser	Kitchen	2,562
Copy Machine	Staff Planning	290
Microwave Oven	Staff Planning	1,366
Fluorescence Light (26W) (94% of its Watt value)	Rooms, Hallways	90
Fluorescence Light (32W) (94% of its Watt value)	Rooms, Hallways	103
400W Light (114% of its Watt value)	Gymnasium	1,557

Infiltration:

Infiltration occurs even there are almost no traffics between in and out of the building. The air can be entering into the building though cracks or gaps in windows, walls or roofs. The building selected has only eleven windows so the infiltrations through crack of window frames are negligible. There are thirty doors but all doors to the outside remain closed during the school hours, thus the air volume change due to traffics during the hours are also very small. The only considerable factor was infiltration due to the cracks between walls and roofs, which were calculated from the size of the room. The air mass of the room and the difference in enthalpy gives total energy difference. The enthalpy for morning, noon and evening to the room were determined from the formula in McQuiston's "*Heating, Ventilating, and Air Conditioning, Fifth Edition*", equation 3-20a (P. 53), the added value due to the water vapor was referred from table A-1a Properties of Refrigerant 718 (Water-Steam) – English unit (P. 586).

Table A-3: Heat Generation due to the Infiltration

Location		V (cfm)	V (ft ³ /hr)	Mass (lb _{air} /hr)	Difference in Btu/lb _{air}			Energy (Btu/hr)		
Zone	Room				Morning	Noon	Evening	Morning	Noon	Evening
1	Kitchen	41.0	2460	5.85	-1	4.2	5.1	-6	25	30
	Vest	15.0	900	2.14	-1	4.2	5.1	-2	9	11
	Maintenance	21.7	1302	3.09	-1	4.2	5.1	-3	13	16
	P.E. Storage	41.0	2460	5.85	-1	4.2	5.1	-6	25	30
	Total	118.7	7122	16.93	-1	4.2	5.1	-17	71	86
2	Gym1	105.5	6330	15.05	-1	4.2	5.1	-15	63	77
3	Gym2	105.5	6330	15.05	-1	4.2	5.1	-15	63	77
4	Hall1	58.0	3480	8.27	-1	4.2	5.1	-8	35	42
	Hall2	0.0	0	0.00	-1	4.2	5.1	0	0	0
	Total	58.0	3480	8.27	-1	4.2	5.1	-8	35	42
5	Staff Bathroom	0.0	0	0.00	-1	4.2	5.1	0	0	0
	Staff Planning	0.0	0	0.00	-1	4.2	5.1	0	0	0
	Music Storage	0.0	0	0.00	-1	4.2	5.1	0	0	0
	Total	0.0	0	0.00	-1	4.2	5.1	0	0	0
6	Music Room	0.0	0	0.00	-1	4.2	5.1	0	0	0
7	Art room	49.0	2940	6.99	-1	4.2	5.1	-7	29	36
	Art Storage	28.0	1680	3.99	-1	4.2	5.1	-4	17	20
	Tech Office	0.0	0	0.00	-1	4.2	5.1	0	0	0
	Boys room	11.0	660	1.57	-1	4.2	5.1	-2	7	8
	Girls room	12.0	720	1.71	-1	4.2	5.1	-2	7	9
	Total	100.0	6000	14.26	-1	4.2	5.1	-14	60	73
All	All Rooms	487.7	29262	69.55	-1	4.2	5.1	-109	458	556

Thermal Conduction Through the Walls and Roofs:

The conduction occurs when there is a temperature difference over a medium. In the morning, the outside air is cooler than that is in the rooms. Thus indoor air loses its energy through walls and roofs. In comparison, the higher temperature at outside during daytime causes energy to move into the rooms. Since the building has good insulation, heat gains from conductions are low. The thermal conductances between rooms are neglected since the temperatures in all rooms will be maintained at the same.

The calculations of the thermal resistance of walls and roofs in different constructions are described as below. The values of thermal resistances are referred from McQuiston's "*Heating, Ventilating, and Air Conditioning, Fifth Edition*", table 5-1a (P.127) for building materials, and 5-2a (P.135) for air.

Walls:

1. Type A (1): (with 15mph wind)

a. Component:

i. Outdoor; Moving Air @15mph; $R = 0.17 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

ii. Wall;

1. 3.625in. Face Brick; $R = 0.44 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

2. 2in. R19 BATT Insulation; $R = 12.0 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

3. 0.75in. Gypsum Board; $R = 0.14 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

4. 7.625in. Concrete; $R = 5.00 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

5. 0.625in. Gypsum Board; $R = 0.11 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

iii. Indoor Air; Still Air, vertical surface with horizontal heat flow;

$R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance

$R = 18.54 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

2. Type A (2): (with 7.5 mph wind)

a. Component:

i. Outdoor; Moving Air @7.5mph; $R = 0.28 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

ii. Wall;

1. 3.625in. Face Brick; $R = 0.44 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

2. 2in. R19 BATT Insulation; $R = 12.0 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

3. 0.75in. Gypsum Board; $R = 0.14 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

4. 7.625in. Concrete; $R = 5.00 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

5. 0.625in. Gypsum Board; $R = 0.11 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

iii. Indoor Air; Still Air, vertical surface with horizontal heat flow;

$R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance

$R = 18.43 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

3. Type B; (Used in Gymnasium Walls)

a. Construction:

i. Outdoor; Moving Air @15mph; $R = 0.28 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
ii. Wall;

1. 3.625in. Face Brick; $R = 0.44 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
2. 2in. Rigid Insulation; $R = 12.0 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
3. 0.75in. Gypsum Board; $R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
4. 11.625in Concrete; $R = 5.8 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

iii. Indoor Air; Still Air, vertical surface with horizontal heat flow;
 $R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance

$$R = 19.84 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$$

4. Type C: (Used in Hallway Wall)

a. Composition:

i. Outdoor; Moving Air @15mph; $R = 0.28 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
ii. Wall;

1. 3.625in. Face Brick; $R = 0.44 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
2. 2in. R19 BATT Insulation; $R = 12.0 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
3. 0.75in. Gypsum Board; $R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
4. 7.625in. Concrete; $R = 5.00 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

iii. Indoor Air; Still Air, vertical surface with horizontal heat flow;
 $R = 0.68 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance

$$R = 18.12 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$$

Roofs:

5. Type A:

a. Composition:

i. Outdoor; Moving Air @15mph; $R = 0.17 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
ii. Roofing;

1. 0.5in. Cover; $R = 0.33 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
2. 4in. Rigid Insulation; $R = 24 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
3. 1in. Gypsum Board; $R = 0.93 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
4. 1.5in. Metal Deck; $R = 0.56 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
5. 0.5in. Acoustic Board; $R = 2.94 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

iii. Indoor Air; Still Air, horizontal surface with upward heat flow;
 $R = 0.61 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance

$$R = 29.54 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$$

6. Type B: (Used for Gymnasium roof)

a. Construction:

- i. Outdoor; Moving Air @15mph; $R = 0.17 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
- ii. Roofing;

- 1. 0.5in. Cover; $R = 0.33 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
- 2. 4in. Rigid Insulation; $R = 24 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$
- 3. 1.5in. Metal Deck; $R = 0.56 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

- iii. Indoor Air; Still Air, horizontal surface with upward heat flow;
 $R = 0.61 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

b. Total Thermal Resistance $R = 25.67 \text{ (hr-ft}^2\text{-}^\circ\text{F/Btu)}$

By knowing the indoor and outdoor temperatures, and the area of the walls and roofs, the amount of heat transfer can be determined.

Table A-5a: Table of Heat Transfer Through the Walls

Location:		Direction:	Type:	A_{wall} (ft ²)	Temp.(F)			R (hr-ft ² -F/Btu)			Q (Btu/hr)			
Zone:	Room:				M	N	E	Wall	Air _{outdoor}	Air _{indoor}	M	N	E	
1	Kitchen	S	A	270	66	80	85	73	17.58	0.28	0.68	-102	102	175
		W	A	252	66	80	85	73	17.58	0.17	0.68	-96	96	164
	Vestibule	W	A	81	66	80	85	73	17.58	0.17	0.68	-31	31	53
	Maint. Strg.	W	A	243	66	80	85	73	17.58	0.17	0.68	-92	92	158
	P.E. Strg.	W	A	144	66	80	85	73	17.58	0.17	0.68	-55	55	94
	Room	N	A	270	66	80	85	73	17.58	0.28	0.68	-102	102	175
2	Gym 1	S	B	1426	66	80	85	73	18.88	0.28	0.68	-503	503	863
3	Gym 2	N	B	1426	66	80	85	73	18.88	0.28	0.68	-503	503	863
4	Hallway 1	W	C	288	66	80	85	73	5.80	0.28	0.68	-298	298	511
		N	A	90	66	80	85	73	17.58	0.28	0.68	-34	34	58
	Hallway 2	-	-	-	-	-	-	-	-	-	-	-	-	-
5	Staff Restroom	-	-	-	-	-	-	-	-	-	-	-	-	-
	Staff Planning	-	-	-	-	-	-	-	-	-	-	-	-	-
	Music Strg.	-	-	-	-	-	-	-	-	-	-	-	-	-
6	Music Room	-	-	-	-	-	-	-	-	-	-	-	-	-
7	Art	N	A	360	66	80	85	73	17.58	0.28	0.68	-136	136	233
	Art Strg. Room	N	A	171	66	80	85	73	17.58	0.28	0.68	-65	65	111
		NE	A	126	66	80	85	73	17.58	0.28	0.68	-48	48	82
	Boys Restroom	NE	A	117	66	80	85	73	17.58	0.28	0.68	-44	44	76
	Girls Restroom	NE	A	117	66	80	85	73	17.58	0.28	0.68	-44	44	76

Table A-5b: Table of Heat Transfer Though the Roofs

Location:		Direction:	Type:	A_{roof} (ft ²)	Temp. (F)				R (hr-ft ² -F/Btu)			Q (Btu/hr)		
Zone:	Room:				M	N	E	R	Roof	Air _{outdoor}	Air _{indoor}	M	N	E
1	Kitchen	Roof	A	826	66	80	85	73	28.76	0.17	0.61	-196	196	336
	Vestibule	Roof	A	95	66	80	85	73	28.76	0.17	0.61	-23	23	39
	Maint. Strg.	Roof	A	909	66	80	85	73	28.76	0.17	0.61	-215	215	369
	P.E. Strg.	Roof	A	460	66	80	85	73	28.76	0.17	0.61	-109	109	187
2	Gym 1	Roof	B	2417	66	80	85	73	24.89	0.17	0.61	-659	659	1130
3	Gym 2	Roof	B	2417	66	80	85	73	24.89	0.17	0.61	-659	659	1130
4	Hallway 1	Roof	A	1296	66	80	85	73	28.76	0.17	0.61	-307	307	526
	Hallway 2	Roof	A	842	66	80	85	73	28.76	0.17	0.61	-200	200	342
5	Staff Plan.	Roof	A	780	66	80	85	73	28.76	0.17	0.61	-185	185	317
	Staff Toilet	Roof	A	52	66	80	85	73	28.76	0.17	0.61	-12	12	21
	Music Storage	Roof	A	222	66	80	85	73	28.76	0.17	0.61	-53	53	90
6	Music Room	Roof	A	1165	66	80	85	73	28.76	0.17	0.61	-276	276	473
7	Art Room	Roof	A	1278	66	80	85	73	28.76	0.17	0.61	-303	303	519
	Art Storage Room	Roof	A	388	66	80	85	73	28.76	0.17	0.61	-92	92	158
	Tech Office	Roof	A	105	66	80	85	73	28.76	0.17	0.61	-25	25	43
	Boys Bathroom	Roof	A	266	66	80	85	73	28.76	0.17	0.61	-63	63	108
	Girls Bathroom	Roof	A	266	66	80	85	73	28.76	0.17	0.61	-63	63	108

Heat Transfer Due to the Sun's Radiation:

The radiation from the Sun is another large portion of heat addition so it must be taken into account. Since the south side of the building complex is attached to other part of the school and other sides are also surrounded by nearby buildings, most of the radiations to the walls are neglected. The amount of radiation receiving from the Sun was reduced by 25% in morning and evening since the weather conditions were partly cloudy, and 50% reduction in the noon since it was mostly cloudy. The roofing of the school has ϵ of 0.1. Here is the calculation to determine the Sun's location and time. The equations of time and its calculation procedure are based on McQuiston's "*Heating, Ventilating, and Air Conditioning, Fifth Edition*", chapter 6, page 160 ~ 168. The declination angle and radiation rate are based on table 6-1 (P. 165)

1. Location of Salina, Kansas: N. 38.80deg. W. 97.65deg.

$$EOT = 229.2 (0.000075 + 0.001868\cos N - 0.032077\sin N - 0.014615\cos 2N - 0.04089\sin 2N) \text{ minutes}$$

Where,

$$N = (n-1)(360/365)$$

n = number of days since Jan. 1st

The date is September 1, 2001 (n = 272) so, N = 267.2876712

Equation of Time = 9.789499 min.

2. Declination Angle (δ)

$$\delta = 0.3963723 - 22.9132745\cos N + 4.0254304\sin N - 0.3872050\cos 2N + 0.05196728\sin 2N - 0.1545267\cos 3N + 0.08479777\sin 3N$$

Where,

$$N = 267.2876712$$

$\delta = 0.0$ (September had declination of 0.0 degrees)

	8:00a.m.,	1:00p.m.,	5:00p.m.
CST=CDST-1 hour =	7:00a.m.,	12:00p.m.,	4:00p.m. (at 90deg.W)
LCT=CST- 4x(97.65-90.00)	6:29a.m.,	11:29a.m.,	3:29p.m. (at 97.65deg W.)
LST=LCT+EOT	6:38a.m.,	11:38a.m.,	3:38p.m. (Local Sun Time)
The hour Angle (h)deg	80.75	6.25	-54.25 (0 degrees at noon)
Solar Altitude: β	7.20	50.78	27.09
Zenith angle: ψ	82.80	39.22	62.91
Solar Azimuth angle: Φ	84.17	9.89	1.83
Wall Solar Azimuth: γ	166.97	49.11	64.74 (1)
Angle of Incidence: θ	165.14	65.55	67.67 (Horizontal1)
	82.8	39.22	62.91 (Vertical)

In September, the average radiation of Sun is 360.2 Btu/hr-ft². The calculations done according to this value and the heat addition to each room has been tabulated.

Table A-6: Heat Addition to the Rooms due to the Radiation

Zone:	Room:	Location:	A _{roof} (ft ²)	Energy Received (Btu/hr)		
				Morning	Noon	Evening
1	Kitchen		826	1466	10698	9012
	Vestibule		95	169	1230	1036
	Maint. Strg.		909	1614	11773	9917
	P.E. Strg.		460	817	5958	5019
2	Gym 1		2417	4291	31304	26369
3	Gym 2		2417	4291	31304	26369
4	Hallway1		1296	2301	16785	14139
	Hallway2		842	1495	10905	9186
5	Staff Planning Room		780	1385	10102	8510
	Staff Restroom		52	92	673	567
	Music Storage Room		222	394	2875	2422
6	Music Room		1165	2068	15089	12710
7	Art Room		1278	2269	16552	13943
	Art Storage Room		388	689	5025	4233
	Tech. Office		105	186	1360	1146
	Boys Bathroom		266	472	3445	2902
	Girls Bathroom		266	472	3445	2902

Detailed Information of the Heat Generated for Different Times

In order to determine the required performance of the air conditioning unit, total cooling load for the each zone must to be known. The values are determined for the hours of analysis, and the original values are from the previous sections.

The zone 1 is consisted of kitchen, vestibule, maintenance and P.E. storage rooms. The kitchen has dining staffs for food preparation in the morning and noon. Most of other rooms will not have people all day, except for a short term.

TableA-7a: Detailed Information of Zone 1

Location		Item	Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
Zone	Room		Morning	Noon	Evening	M	N	E	M	N	E
1	Kitchen	32W light (9)	927	927	927	927	927	927	0	0	0
		Adults (4/4/0)	5,800	5,800	0	2,320	2,320	0	3,480	3,480	0
		Children (0/10/0)	0	6,000	0	0	3,300	0	0	2,700	0
		Freezer (2)	3,680	3,680	3,680	3,680	3,680	3,680	0	0	0
		Hot Box (2)	5,380	5,380	0	5,380	5,380	0	0	0	0
		Hot Bar (1)	990	990	0	990	990	0	0	0	0
		Cold Bar (2)	990	990	0	990	990	0	0	0	0
		Beverage Dispenser (1)	2,562	2,562	2,562	2,562	2,562	2,562	0	0	0
		Dish washer (2)	0	190	0	0	190	0	0	0	0
		Infiltration	-6	25	30	-6	12	15	0	12	15
		Wall (Direct)	-198	198	339	-198	198	339	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-196	196	336	-196	196	336	0	0	0
		Roof (Indirect)	1,473	10,769	8,951	1,473	10,769	8,951	0	0	0
		Total	21,403	37,705	16,824	17,923	31,513	16,809	3,480	6,192	15
Vest		32W light (1)	103	103	0	103	103	0	0	0	0
		Adults (1/1/0)	740	740	0	370	370	0	370	370	0
		Infiltration	-2	9	11	-2	4	5	0	4	5
		Wall (Direct)	-31	31	53	-31	31	53	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-23	23	39	-23	23	39	0	0	0
		Roof (Indirect)	169	1,239	1,029	169	1,239	1,029	0	0	0
		Total	957	2,145	1,132	587	1,770	1,127	370	374	5
Maint. Strg.		32W lights (10)	1,034	1,034	0	1,034	1,034	0	0	0	0
		Infiltration	-3	13	16	-3	6	8	0	6	8
		Wall (Direct)	-92	92	158	-92	92	158	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-215	215	369	-215	215	369	0	0	0
		Roof (Indirect)	1,621	11,851	9,850	1,621	11,851	9,850	0	0	0
		Total	2,344	13,205	10,393	2,344	13,199	10,385	0	6	8
P.E. Storage		32W lights (6)	620	620	0	620	620	0	0	0	0
		Adults (4/0/0)	5,800	0	0	2,320	0	0	3,480	0	0
		Infiltration	-6	25	30	-6	12	15	0	12	15
		Wall (Direct)	-157	157	269	-157	157	269	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-109	109	187	-109	109	187	0	0	0
		Roof (Indirect)	820	5,997	4,985	820	5,997	4,985	0	0	0
		Total	6,969	6,908	5,470	3,489	6,895	5,455	3,480	12	15

Zone 2 and 3 are both in the gymnasium. The facility will be used for both exercise and dining. During noon, students will be eating in the gym about 150 students at a time on the average.

Table A-7b: Detailed Information of Zone 2 and 3

Location		Item	Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
Zone	Room		Morning	Noon	Evening	M	N	E	M	N	E
2	Gym1	32W lights (6)	620	620	620	620	620	620	0	0	0
		400W lights (6)	9,343	9,343	0	9,343	9,343	0	0	0	0
		Adults (3/0/0)	4,350	0	0	1,740	0	0	2,610	0	0
		Children (0/75/0)	0	30,000	0	0	16,500	0	0	13,500	0
		Infiltration	-15	63	77	-15	32	38	0	32	38
		Wall (Direct)	-503	503	863	-503	503	863	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-659	659	1,130	-659	659	1,130	0	0	0
		Roof (Indirect)	4,310	31,510	26,190	4,310	31,510	26,190	0	0	0
		Total	17,446	72,700	28,880	14,836	59,168	28,842	2,610	13,532	38
3	Gym2	32W lights (6)	620	620	620	620	620	620	0	0	0
		400W lights (6)	9,343	9,343	0	9,343	9,343	0	0	0	0
		Adults (3/0/0)	4,350	0	0	1,740	0	0	2,610	0	0
		Children (0/75/0)	0	30,000	0	0	16,500	0	0	13,500	0
		Infiltration	-15	63	77	-15	32	38	0	32	38
		Wall (Direct)	-503	503	863	-503	503	863	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-659	659	1,130	-659	659	1,130	0	0	0
		Roof (Indirect)	4,310	31,510	26,190	4,310	31,510	26,190	0	0	0
		Total	17,446	72,700	28,880	14,836	59,168	28,842	2,610	13,532	38

The both hallways are considered as one zone since they are connected with open space. In the morning and evening, there will be no student in the building. During the noon, about 10 ~ 30 people are walking on the hallways on the average.

Table A-7c: Detailed Information of Zone 4

Zone	Location	Item	Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
			Morning	Noon	Evening	M	N	E	M	N	E
4	Hall1	32W lights (10)	1,034	1,034	1,034	1,034	1,034	1,034	0	0	0
		Adults (3/3/2)	2,220	2,220	1,480	1,110	1,110	740	1,110	1,110	740
		Children (0/7/0)	0	4,200	0	0	2,100	0	0	2,100	0
		Infiltration	-8	35	42	-8	17	21	0	17	21
		Wall (Direct)	-332	332	569	-332	332	569	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-307	307	526	-307	307	526	0	0	0
		Roof (Indirect)	2,311	16,896	14,043	2,311	16,896	14,043	0	0	0
		Total	4,917	25,024	17,696	3,807	21,797	16,934	1,110	3,227	761
		32W lights (9)	990	990	990	990	990	990	0	0	0
	Hall2	Adults (3/3/3)	2,220	2,220	2,220	1,110	1,110	1,110	1,110	1,110	1,110
		Children (0/7/0)	0	4,200	0	0	2,100	0	0	2,100	0
		Infiltration	0	0	0	0	0	0	0	0	0
		Wall (Direct)	0	0	0	0	0	0	0	0	0
		Wall (Indirect)	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-200	200	342	-200	200	342	0	0	0
		Roof (Indirect)	1,501	10,977	9,124	1,501	10,977	9,124	0	0	0
		Total	4,512	18,587	12,676	3,402	15,377	11,566	1,110	3,210	1,110

For zone 5, the air conditioning unit providing air to the staff restroom and planning room, and also music storage room. Very low heat generations are expected over the day.

Table A-7d: Detailed Information of Zone 5

Location		Item	Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
Zone	Room		Morning	Noon	Evening	M	N	E	M	N	E
5	Staff Restroom	32W lights (2)	220	220	220	220	220	220	0	0	0
		Hand Dryer (1)	1,366	1,366	0	1,366	1,366	0	0	0	0
		Adult (1/1/0)	740	740	0	370	370	0	370	370	0
		Infiltration	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-12	12	21	-12	12	21	0	0	0
		Roof (Indirect)	93	678	563	93	678	563	0	0	0
		Total	2,406	3,016	805	2,036	2,646	805	370	370	0
	Staff Planning Room	32W lights (9)	990	990	0	990	990	0	0	0	0
		Adults (10/15/0)	5,900	8,850	0	3,245	4,868	0	2,655	3,983	0
		Infiltration	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-185	185	317	-185	185	317	0	0	0
		Roof (Indirect)	1,391	10,169	8,452	1,391	10,169	8,452	0	0	0
		Total	8,096	20,194	8,769	5,441	16,211	8,769	2,655	3,983	0
	Music Storage Room	32W lights (3)	330	330	330	330	330	330	0	0	0
		Infiltration	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-53	53	90	-53	53	90	0	0	0
		Roof (Indirect)	396	4,341	2,406	396	4,341	2,406	0	0	0
		Total	673	4,724	2,826	673	4,724	2,826	0	0	0

The zone 6 is for music room only. The music instruments need to be maintained under specific condition. The room will be used during the noon.

Table A-7e: Detained Information of Zone 6

Location		Item	Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
Zone	Room		Morning	Noon	Evening	M	N	E	M	N	E
6	Music Room	32W lights (15)	1,650	1,650	1,650	1,650	1,650	1,650	0	0	0
		Adults (0/1/0)	0	445	0	0	267	0	0	178	0
		Children (0/20/0)	0	7,200	0	0	4,320	0	0	2,880	0
		Computers (5)	6,830	6,830	0	6,830	6,830	0	0	0	0
		Infiltration	0	0	0	0	0	0	0	0	0
		Roof (Direct)	-276	276	473	-276	276	473	0	0	0
		Roof (Indirect)	2,077	22,782	12,624	2,077	22,782	12,624	0	0	0
		Total	10,281	39,183	14,747	10,281	36,125	14,747	0	3,058	0

This zone covers most of the east side of the building. Art room will be occupied during the noon, and tech office will be staffed most of the day. Both restrooms are expected to be occupied largely during the noon hour.

Summary of the Cooling Load

By adding the all heat generated / added by occupants, appliances, infiltration, conduction and radiation, the cooling loads for each room and zone can be summarized.

Table A-8: Summary of the Cooling Load

Location:		Heat Generated (Btu/hr)			Sensible Heat (Btu/hr)			Latent Heat (Btu/hr)		
Zone	Room	Morning	Noon	Evening	Morning	Noon	Evening	Morning	Noon	Evening
1	Kitchen	21,403	37,705	16,824	17,923	31,513	16,809	3,480	6,192	15
	Vest	957	2,145	1,132	587	1,770	1,127	370	374	5
	Maintenance	2,344	13,205	10,393	2,344	13,199	10,385	0	6	8
	P.E. Storage	6,969	6,908	5,470	3,489	6,895	5,455	3,480	12	15
	Total	31,673	59,963	33,819	24,343	53,378	33,775	7,330	6,586	43
2	Gym1	17,446	72,700	28,880	14,836	59,168	28,842	2,610	13,532	38
3	Gym2	17,446	72,700	28,880	14,836	59,168	28,842	2,610	13,532	38
4	Hallway1	4,917	25,024	17,696	3,807	21,797	16,934	1,110	3,227	761
	Hallway2	4,512	18,587	12,676	3,402	15,377	11,566	1,110	3,210	1,110
	Total	9,429	43,611	30,371	7,209	37,173	28,500	2,220	6,437	1,871
5	Staff Restroom	2,406	3,016	805	2,036	2,646	805	370	370	0
	Staff Planning	8,096	20,194	8,769	5,441	16,211	8,769	2,655	3,983	0
	Music Storage	673	4,724	2,826	673	4,724	2,826	0	0	0
	Total	11,176	27,934	12,399	8,151	23,581	12,399	3,025	4,353	0
6	Music Room	10,281	39,183	14,747	10,281	36,125	14,747	0	3,058	0
7	Art room	10,493	33,434	16,466	10,493	30,362	16,448	0	3,073	18
	Art Storage	814	5,610	4,575	814	5,602	4,565	0	8	10
	Tech Office	972	2,888	1,400	707	2,623	1,400	266	266	0
	Boys room	696	7,078	3,074	696	6,174	3,070	0	903	4
	Girls room	805	7,188	3,075	805	6,285	3,070	0	904	4
	Total	13,780	56,199	28,591	13,515	51,045	28,554	266	5,153	36
All	All Area	111,231	372,289	177,687	93,171	319,639	175,660	18,061	52,650	2,027

Air Circulation and the Selection of the Unit:

Certain amount of air must be delivered to each room in order to maintain the room temperature and moisture level. In addition, enforcing the health regulations regarding to fresh air requirement set by the government is also an important factor.

Fresh Air Requirement:

For different conditions of the room, there are certain amount of air which have to be replaced with outside air. The designed system must be able to deliver enough fresh air to each room in the zone. These numbers must be balanced carefully since too much fresh air drive the operation cost up, and lower amount of air can causes health problem to the occupants. For this project, the amount of fresh air should not exceed more than 50%, except for the gym where has fresh air ratio of 80%, simply because of very high level of activity is expected. Also for the restrooms, re-circulation of air is not recommended. The number of the occupants on the determined from averaged number of the people who are in the rooms.

Table A-9: Chart of Fresh Air Requirement for Each Room

Zone	Room	A (ft ²)	Vol. (ft ³)	Air Requirement		# of Occupants			Fresh Air Needed (cfm)		
				Value	Unit	M	N	E	M	N	E
1	Kitchen	826	7434	30 cfm/occu.	4		14 T	0	120	420	0
	Vest	95	855	1 cfm/ft ²	1	T/O	1 T/O	0	95	95	95
	Maint. Strg.	909	8181	0.33 cfm/ft ²	0		0	0	300	300	300
	P.E. Strg.	460	4140	0.33 cfm/ft ²	4	T	0	0	152	152	152
	Total								667	967	547
2	Gym1	2417	55591	50 cfm/occu.	3		75 T	0	150	3750	0
3	Gym2	2417	55591	50 cfm/occu.	3		75 T	0	150	3750	0
4	Hall1	1296	11664	0.33 cfm/ft ²	3		10	2	428	428	428
	Hall2	842	7578	0.33 cfm/ft ²	3		10	3	278	278	278
	Total								706	706	706
5	Staff Bath	52	468	43 cfm/toilet	1 T	1 T	0		43	43	43
	Staff Planning	780	7020	20 cfm/occu.	10 T	15 T	0		200	300	0
	Music Strg.	222	1998	1 cfm/ft ²	0		0	0	222	222	222
	Total								465	565	265
6	Music	1165	10485	20 cfm/occu.	0		21	0	0	420	0
7	Art Room	1278	11502	20 cfm/occu.	0		21	0	0	420	0
	Art Strg.	388	3492	1 cfm/ft ²	0		0	0	388	388	388
	Tech Office	105	945	20 cfm/occu.	1		1	0	20	20	0
	Boys room	266	2394	30 cfm/occu	0		3 T	0	90	90	90
	Girls room	266	2394	35 cfm/toilet	0		3 T	0	90	90	90
	Total								588	1008	568

T indicates traffic, and T/O is for traffic to the outside

Ratio of the fresh air

In order to satisfy the fresh air requirement, the combination of the fresh and returned air must be carefully determined. Once the percentage of fresh air for the unit is chosen, it is not possible to change the ratio by the room, thus the air volume delivered to each room must satisfy both air supply and fresh air requirement at design phase. The unit is allowed to have fresh air ratio of between 20 to 50% to maintain low operating cost, except for the gymnasium where high level of activities are expected. The fresh air ratios for the zones are chosen based on the considerations above.

Table A-10: Air Supply / Fresh Air Ratio for Zones

Location:		Capacity (ton)	Air _{Supply} (cfm)	Air _{Fresh} (cfm)	%Air _{Fresh}
Zone	Room				
1	Kitchen		1,904	420	22%
	Vest		190	95	50%
	Maintenance		667	300	45%
	P.E. Storage		352	152	43%
	Total	5	3,100		50%
2	Gym1	6	4,700	3750	80%
3	Gym2	6	4,700	3750	80%
4	Hallway1		1,426	428	30%
	Hallway2		939	278	30%
	Total	4	2,365		30%
5	Staff Restroom		152	43	28%
	Staff Planning Room		1,020	300	29%
	Music Storage		444	222	50%
	Total	2	1,616		50%
6	Music Room	3	1,979	420	21%
7	Art room		1,689	420	25%
	Art Storage		776	388	50%
	Tech Office		146	20	14%
	Boys room		200	100	50%
	Girls room		200	100	50%
	Total	5	3,011	1028	50%
All	All Area	31	18,802		

Selection of the Air Conditioning Unit

The units for each zone were selected based on the cooling capacity and the ability of airflow delivery rate required. To determine the cooling load, table A-8 was used and for the air supply, table A-10 was used as reference. Refer making on the plan "RTU" with a number for each zone.

Table A-11: Marking on the plan and the Unit Information

Mark	Capacity (ton)	Air _{Supply} (cfm)	Model#
RTU1	5	3,100	L6A060
RTU2	6	4,700	L6A100
RTU3	6	4,700	L6A100
RTU4	4	2,365	L6A048
RTU5	2	1,616	L6A036
RTU6	3	1,979	L6A036
RTU7	4	3,011	L6A060

Notes:

1. Job site elevation is 1,300ft.
2. External Static Pressure includes economizer and 0.5in.w.g. for dirty filter.
3. Provide manufacturer's recommended service clearance around entire unit.
4. Heating and Cooling capacities and fan static pressures are at jobsite altitude.
5. Provide 2in., 30% efficient filters.
6. Provide full perimeter roof curb.
7. Unit to have single point electrical connection.
8. Provide factory installed non-fused disconnect.
9. Provide with thru the base electrical connection.

The information of the air conditioning units are available on the back of this report.

Estimate of Cost of the HVAC system:

1. Duct work

Type	in ³ of ductwork	Cost/lb	Total Cost
Medium Pressure	8310	3.57*	24699**

*Price sheet is attached to document

**Costs include fabrication, field assembly, labor and installation per pound of duct installed

2. Fans

Exhaust Size	# of units	inlet size	Labor/unit	Cost
250 cfm	2	8"x8"	\$ 90.00	\$ 500.00
1200 cfm	1	12"x12"	\$ 90.00	\$ 300.00

3. Rooftop Units

Zone	Rooftop type	# of units	cfm	Unit Cost	Labor	Total Cost
1	Single Zone	1	3100	\$3,500.00	\$ 450.00	\$5,500.00
2	Single Zone	1	4700	\$6,100.00	\$ 500.00	\$7,000.00
3	Single Zone	1	4700	\$6,100.00	\$ 500.00	\$7,000.00
4	Single Zone	1	2365	\$3,100.00	\$ 450.00	\$3,550.00
5	Single Zone	1	1616	\$2,800.00	\$ 450.00	\$4,800.00
6	Single Zone	1	1979	\$3,000.00	\$ 450.00	\$5,000.00
7	Single Zone	1	3011	\$3,200.00	\$ 450.00	\$5,200.00

4. Fire Dumpers

# of units	Gauge	Labor	Unit Cost	Total Cost
6	14	12.20/ea	\$ 26.20	\$ 230.40

5. Grills and Diffusers

Type	# of Units	Unit Costs	Size	Equipment/unit	Labor/unit	Total Cost
Supply	35	80	24"x24"	\$ 0.14	\$ 8.30	\$3,095.40
Return	10	48	24"x24"	\$ 0.14	\$ 14.30	\$ 624.40
Gym	8	40.9	18"x8"	\$ 0.14	\$ 8.30	\$ 394.72

6. Filters

Type	# of Units	Unit Costs	Equipment	Labor/unit	Total Cost
99.99% Efficiency	7	\$ 262.00	\$ 0.88	\$ 51.50	\$2,200.66

7. Turning Valves

Type	# of Units	Unit Costs	Equipment	Labor/unit	Total
Elbow	5	\$ 5.60	\$ 0.02	\$ 1.68	\$ 36.50

8. Cost

- a. Total Cost \$70,131.08
- b. Multiplier for construction x1.5
- c. Total Estimated Construction Costs \$105,196.60
- d. Building Estimated Construction Costs \$8,000,000
- e. HVAC design Estimation Cost \$80,000

Heavy Duty Gym Grilles 90 / 91 / 95 / 96 Series



Product Information

Heavy Gauge Steel Models

3/8" (10) Blade Spacing, 0° Deflection
Grille 90
Register w/ Steel Damper 90D
3/8" (10) Blade Spacing, 45° Deflection
Grille 91
Register w/ Steel Damper 91D
3/4" (19) Blade Spacing, 0° Deflection
Grille 95
Register w/ Steel Damper 95D
3/4" (19) Blade Spacing, 45° Deflection
Grille 96
Register w/ Steel Damper 96D

Application:

- Specifically designed for severe applications such as gymnasiums, factories, warehouses, public washrooms, heavy traffic corridors and any general public area.

Construction:

- 4 Core styles available of heavy 14 gauge steel.
- Heavy gauge 1 1/4" (32) flat steel border. Smooth contours reduce the possibility of injury to athletes when used in a gymnasium.
- Optional opposed blade damper in steel construction with black finish.

Mounting / Finish Options

Blade Orientation

Blades parallel to Long Dimension	L
Blades parallel to Short Dimension	S

Optional Mounting Frame

Fastening	D
Counter sunk screwholes c/w oval-head screws	A
Concealed (n/a on Model 91)	C

Finish

White	B12
Optional finishes – contact Price representative.	SPL

Available Sizes

Minimum	Maximum*
6" x 4" (152 x 102)	48" x 48" (1219 x 1219)

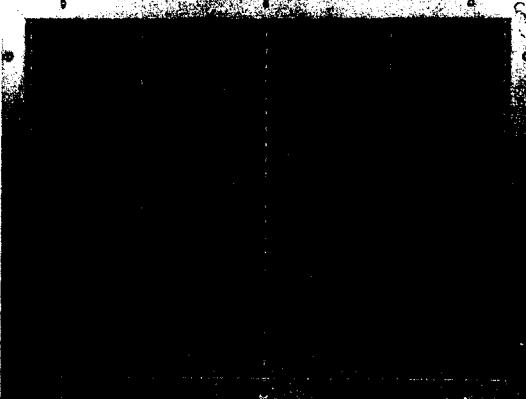
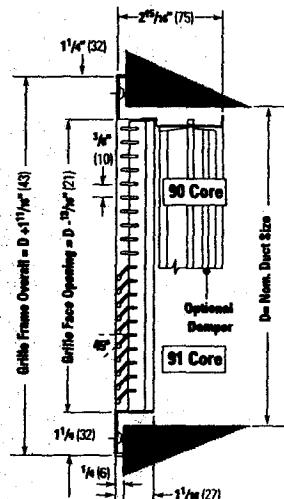
* One piece

Product Information Index

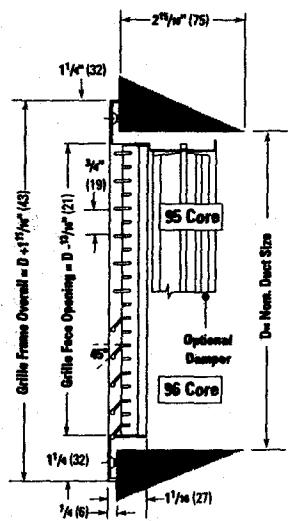
Performance Data D25 - D28
Suggested Specification D112



Model 91



Model 96



✓ Product Selection Checklist

- 1] Select Unit Size based on desired performance characteristics.
- 2] Select Outlet type by model number (Core style, damper).
- 3] Select Blade Orientation.
- 4] Select Mounting Frame if desired.
- 5] Select Fastening type (**A** is standard).
- 6] Select Finish.

Example: 24" x 24" / 95D / L / A / B12

■ Perforated Diffusers – Supply PDC Series

c / w Individually Adjustable Curved Blades



Product Information

Models

Adjustable Curved Blades

Flush Face

Steel Construction
w/ Aluminum Face

PDC
APDC

Drop (Extended) Face

Steel Construction
w/ Aluminum Face

PDCE
APDCE

PRICE PDC / PDCE Series perforated face ceiling diffusers feature individually adjustable curved blades at the diffuser inlet, available in fixed 1, 2, 2-way corner, 3 and 4-way air patterns. The adjustable curved blades provide total flexibility in pattern adjustment from horizontal to vertical as well as low pressure loss and noise levels. Quick release latches on the hinged perforated face screen allow easy access for field adjustment of the blades. The drop (extended) face Model PDCE is available to complement regular tile ceilings. For Fire-Rated applications please see Section G.

Features

- Choice of cold rolled steel (PDC / PDCE) or aluminum face screen (APDC / APDCE) construction. Steel backpan and pattern controllers in both cases.
- Five air pattern options are available from 1 to 4-way.
- Hinged, removable perforated face screen with quick-release spring latches.
- Individually pivoting curved blades are located at the inlet neck and are field adjustable from horizontal to vertical air pattern.
- Choice of five frame styles. (PDCE and APDCE are available in Frame Style 3 only.)
- Complete range of available accessory dampers, equalizing grids etc.

Application

- PDC diffusers are designed for spaces requiring low noise and air pattern adjustment from horizontal to vertical.

Finish

White

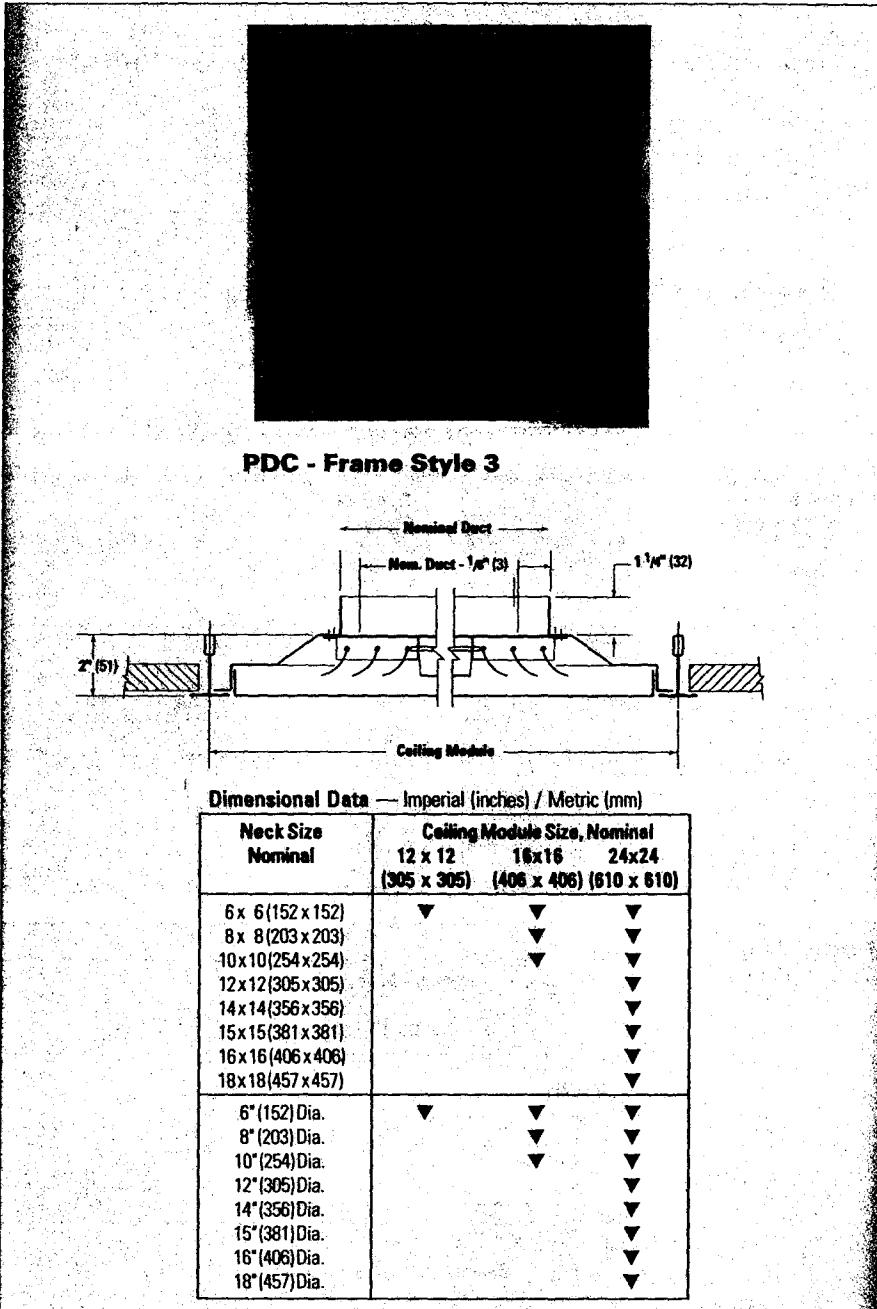
B12

Optional Finishes –
contact Price representative.

SPL

Product Information Index

Frame Selection	C66
Air Pattern Adjustment	C66
Performance Data	C67 - C69
Suggested Specification	C150



CEILING DIFFUSERS

✓ Product Selection Checklist

- 1] Select Inlet diameter or Neck Size L x W based on desired performance characteristics.
- 2] Select Face Size based on ceiling module.
- 3] Select Outlet type by model number (Material, Flush or Drop Face).
- 4] Select Border style according to installation requirements.
- 5] Select Air Pattern Option (1, 2, 2C, 3 or 4-way).
- 6] Select Volume Control accessories, if desired.
- 7] Select Finish.
Example: 8" / 24" x 24" / PDC / 3 / 4 / B12

All Metric dimensions () are soft conversion.

Imperial dimensions are converted to metric and rounded to the nearest millimetre.

■ Perforated Face 10 Series



Product Information

Model

Grille

Steel Core Construction **10**
Aluminum Core Construction **10A**

Application:

- Mid to high capacity return with core free area of approximately 50%.
- Perforated core blends unobtrusively with most interior design conditions.
- Ideally suited for ceiling mounted return applications.

Construction:

- Available in steel or aluminum core construction. (Aluminum core recommended for high humidity applications.)
- Extruded aluminum borders for clean, crisp detailing.
- Core $\frac{3}{16}$ " (8) holes on $\frac{1}{4}$ " (6) centers staggered 60°.
- Mounting frames available to accent grille border and protect wall surface during grille removal.

For Fire-Rated applications please see Section G.

Mounting / Finish Options

Border Style

Surface Mount **F**
 $1\frac{1}{4}$ " (32) Flat (Standard)
T-Bar Lay-In Inverted 1" (25) Tee **TB**

Optional Mounting Frame For F Border

D

Fastening

Counter sunk screwholes
c/w oval-head screws
No screwholes

A

B

Finish

White **B12**
Optional finishes –
contact Price representative.

Available Sizes

Minimum	Maximum*
6" x 4" (152 x 102)	48" x 24" (1219 x 610)

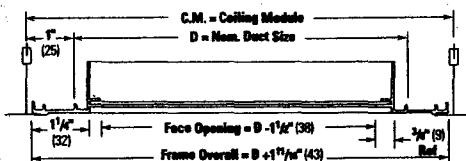
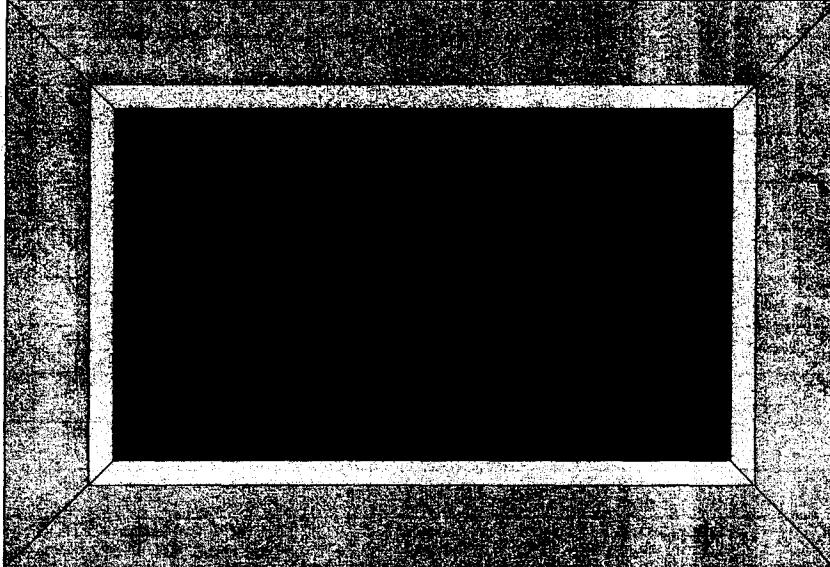
* One piece

- For oversize grille construction,
see page D105-D106.

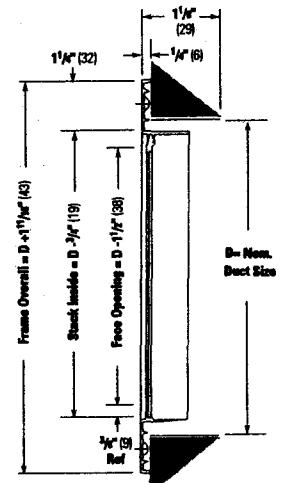
Product Information Index

Performance Data	D21
Available Borders / Frames ..	D58 - D59
Available Fastenings	D58 - D59
Accessory Dampers	D58
Suggested Specification	D110

Perforated Face Return



Lay-In TB Border



Surface Mount F Border

✓ Product Selection Checklist

- 1] Select Unit Size based on desired performance characteristics.
- 2] Select Outlet type by model number (Core style).
- 3] Select Border style according to installation requirements (**F** is standard).
- 4] Select Module Size if TB Border selected. (Note: Maximum duct size = module size-2" (50))
- 5] Select Mounting frame if desired.
- 6] Select Fastening type (**A** is standard, n/a with TB Border).
- 7] Select Finish.

Example: **24" x 12" / 10 / F / A / B12**
22" x 22" / 10A / TB / 24" x 24" / B12

Application Recommendations:

Surface Mount – **F** Border, **A** Fastening.
T-Bar Lay-In – **TB** Border, Fastening n/a

- Fan housing constructed of corrosion resistant galvanized steel.
- Fan housing interior is lined with sound absorbing insulation for quiet operation.
- Grilles are white and attach to fan housing with two screws. Grilles for sizes 210-228 are made of high impact polystyrene. Grilles for sizes 250-265 are constructed of aluminum, coated with baked enamel. All finishes are non-yellowing.
- Outlet duct connection with integral backdraft damper can be converted from horizontal to vertical discharge.
- Fan scroll is constructed of galvanized steel.
- Fan wheels are double width forward curved centrifugal type in a single scroll for SP 210-258. SP 260-265 have twin double width forward curved wheels in separate scrolls driven by a single motor. All wheels are dynamically balanced for vibration free operation.
- Motors are 115/60/1 with built in thermal overload protection, sized to match fan loads and mounted on vibration isolators. Motors are compatible for use with speed controls. Power assemblies can be easily unplugged and removed for inspection or service.
- Angle mounting brackets can be adjusted to any typical ceiling material thickness.

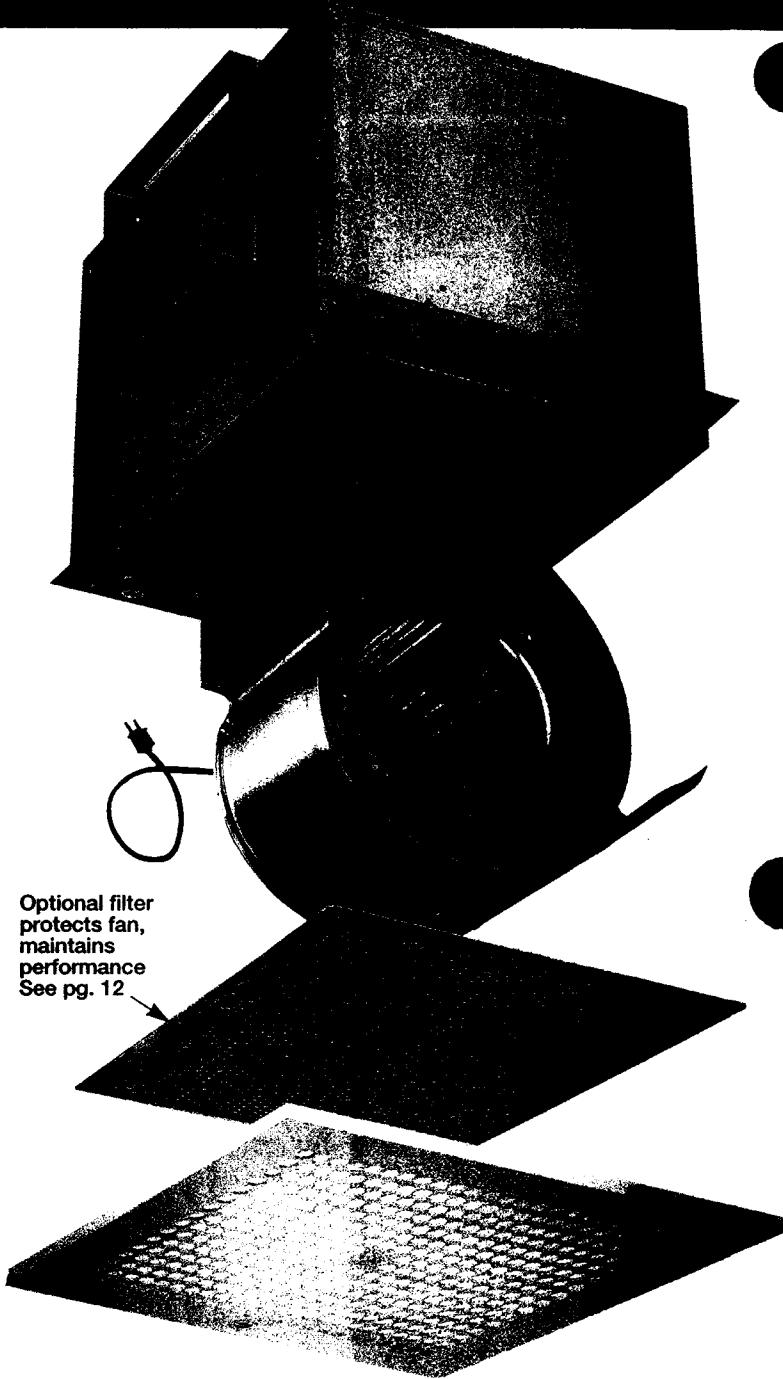
Exclusive Electrical Wiring Feature

- A. Greenheck SP models are the only fans of this type with an electrical access cover located on the housing exterior. This feature permits external wiring without removing the power assembly, saving installation time and cost.



Vertical Discharge

- B. Photo shows exhaust duct adapter installed in the optional vertical position. The power assembly must be rotated to match the duct adapter position.



Round Duct Connector



The model RDC round duct connector replaces the standard outlet for use with round ductwork.

The RDC does not include a damper. The backdraft damper included in the discharge accessory is typically adequate.

The RDC must be specified as an accessory item and ships loose for field installation. See page 14 for details.

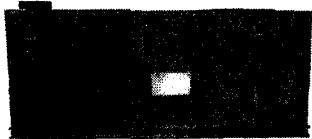
PRODUCT MATRIX

Cabinet	Tonnage	Model	Cooling Efficiency		
			LGA/LCA		LHA
			High	Standard	High
A Box	3	036	12	10	N/A
	3.5	042	12	10	N/A
	4	048	12	10	N/A
	5	060	12	10	N/A
	6	072	10.5	9	N/A
A+ Box	7.5	088	10.2	9	9
	8.5	100	N/A	9	N/A
B Box	8.5	102	11	9	N/A
	10	120	11	9	10.3
	12.5	150	N/A	9	N/A
C Box	13	156	11.5	N/A	N/A
	15	180	11.5	9.2	10
	17.5	210	11.2	9	N/A
	20	240	11	9	10.5
D Box	25	300	10	9	N/A
	30	360	10	N/A	N/A

LENNOX ADVANTAGE

LENNOX L SERIES® VS. CARRIER 48TJ & CARRIER 48HJ

Lennox L Series



A Box

Carrier 48HJ



Feature	Lennox					Carrier					Advantage	
Cap., Tons	3	3.5	4	5	6	3	3.5	4	5	6		
ARI Standard Efficiency												
Lennox L Series - A Box												
EER/SEER	-/10	-/10	-/10	-/10	9.0/-	-/10	N/A	-/10	-/10	9.0/-		
Cap., Btuh	36,000	42,000	48,000	57,500	72,000	35,000	-	47,000	57,000	72,000		
S/T Ratio	0.71	0.70	0.71	0.72	0.72	0.71	-	0.70	0.74	0.70		
Cabinet	82 cu.ft.	79 cu. ft.	-	79 cu. ft.	79 cu. ft.	79 cu. ft.						

ARI High-Efficiency

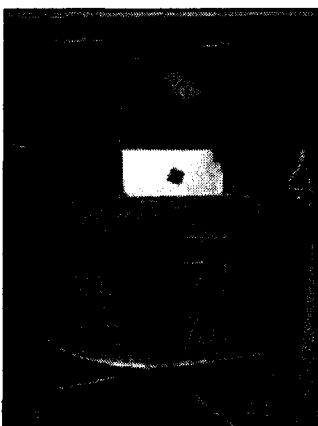
	Lennox L Series - A Box					Carrier 48HJ Model					
EER/SEER	-/12	-/12	-/12	-/12	10.5/-	-/13	N/A	-/13	-/13	11/-	
Cap., Btuh	35,800	42,500	48,000	60,000	71,500	36,000	-	47,000	60,000	74,000	
S/T Ratio	0.73	0.74	0.70	0.73	0.73	0.72	-	0.69	0.70	0.72	
Cabinet	82 cu.ft.	82 cu.ft.	82 cu.ft.	82 cu.ft.	82 cu.ft.	79 cu. ft.	-	79 cu. ft.	79 cu. ft.	79 cu. ft.	

Features

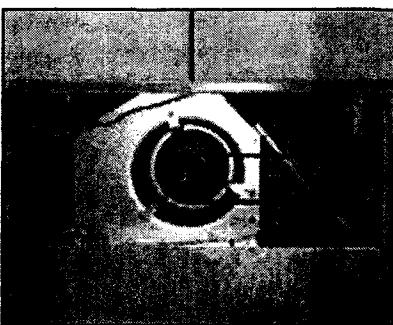
Controls	IMC – Fully integrated with virtually unlimited diagnostics. Includes low ambient and low refrigerant pressure functions. Compatible with most energy management systems and electromechanical thermostats.	IGC – gas units.	Lennox
115 Volt GFI Outlet	15 amp field wired.	Service option package – 115V.	
Factory Installed Options	Disconnect switch with weather proof cover, gear driven economizer and gravity exhaust, smoke detectors, high-efficiency blower motors, coil corrosion protection, dirty filter switch, service valves and DDC control modules. Also available, Humiditrol™ dehumidification.	Apollo DDC, service packages "A" or "H" with the following: GFI, disconnect, RA smoke detector, firestat, hinged access panels. Also available, MoistureMiser™ dehumidification.	Lennox
Utility Connections	Gas = side only; Electric = bottom or side entry. Optional bottom gas thru the curb.	Gas = side only; Electric = bottom or side entry.	
Service Access	Standard toolless, hinged access doors.	Available with optional Service Option Packages.	Lennox
Std. Low Ambient	0° F standard.	25° F standard.	Lennox
Freeze Protection	Standard.	Acutrol.	
Thermostatic Expansion Valve	Standard, with removable power element.	Not Available.	Lennox
Latent Control Option	1 row coil downstream of evaporator. Uses hot gas from compressor to reheat supply air. Humiditrol provides dehumidification on demand.	1 row coil downstream of evaporator Uses hot liquid from condenser coil to reheat supply air. Cooling demand is required to initiate MoistureMiser operation.	Lennox

Feature Comparison

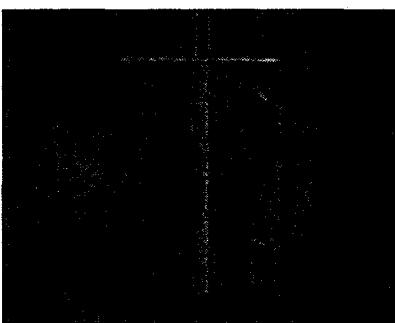
LENNOX ADVANTAGES



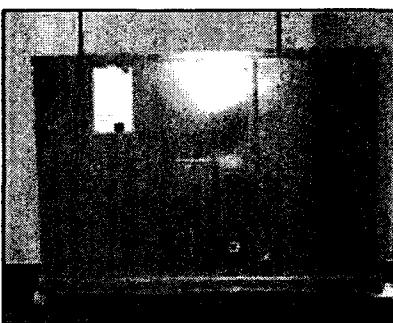
The motor drive pulley is fully accessible for easy adjustment. Two inch commercial grade pleated filters.



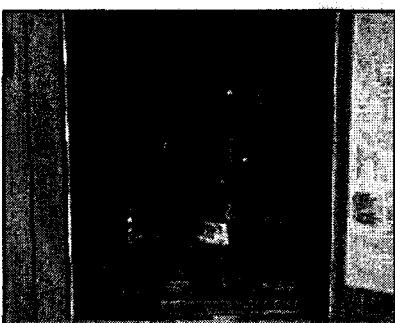
Rigidity of motor mounting is weak and deflects easily as shown in the photo. Access to indoor coil is difficult. Standard two inch unpleated fiberglass filters.



Lennox uses heavy duty galvanized steel hinges with 1/8" brass pins on toolless access panels.



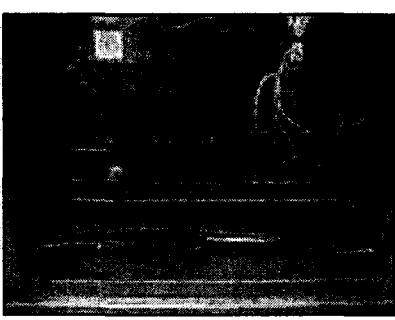
With the exception of the filter access, all other service require removing sheet metal screws to remove cabinet panels. Removed panels are subject to wind-blown damage.



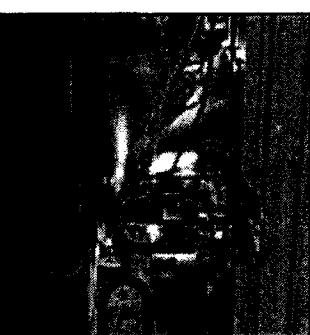
Compressors are easily accessible for testing and service. Independent and isolated compartment facilitates accurate checking of refrigerant charge. TXV's are standard on all L Series units.



In the Carrier unit, access to all refrigeration service ports and filter-dryers are behind the compressors. Most units do not have TXV's. Open compressor-condenser compartment causes difficulty when checking refrigerant charge.

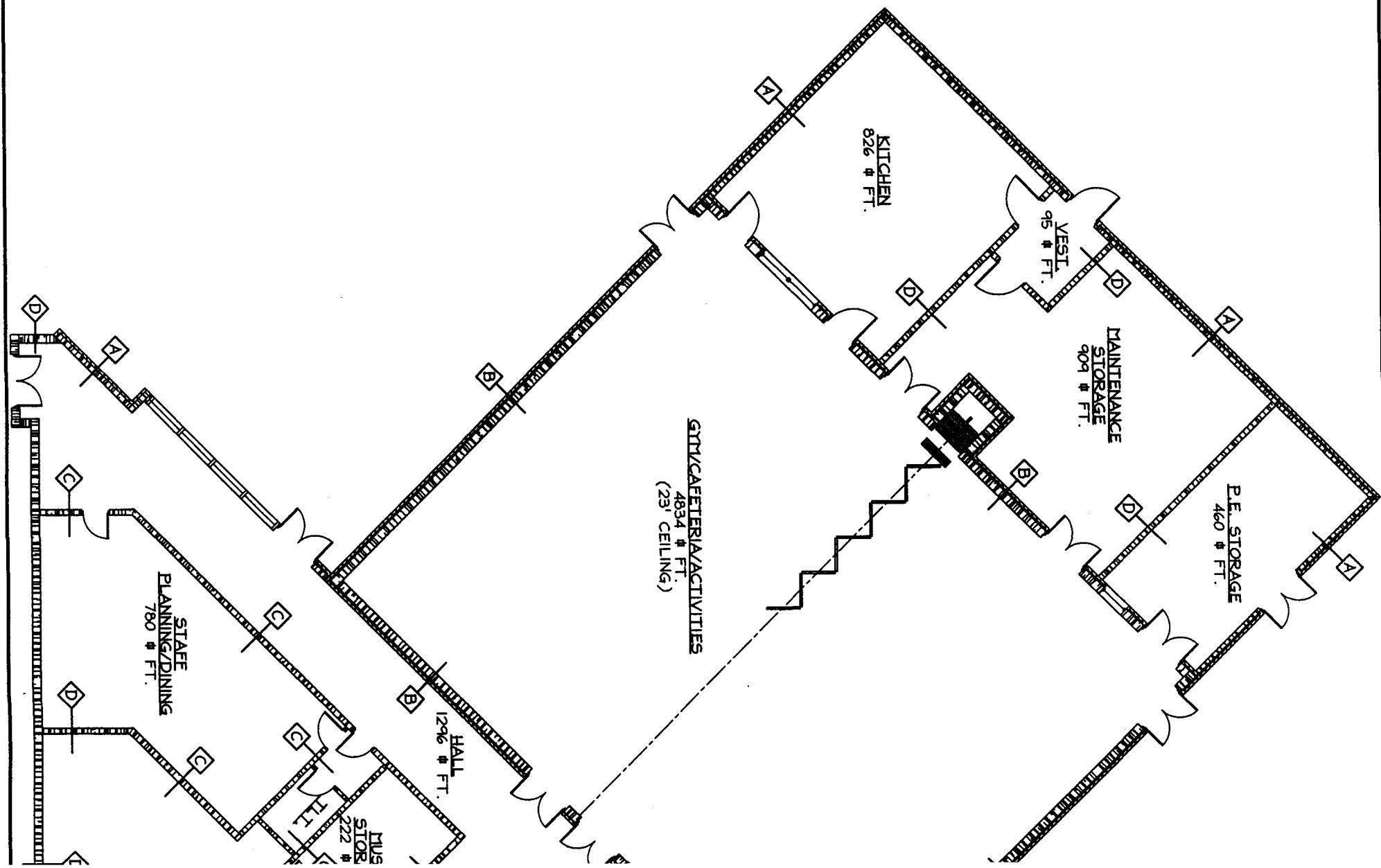


Gas heat section is fully accessible for service, with a slide-out heat exchanger. Optional stainless steel heat exchanger.



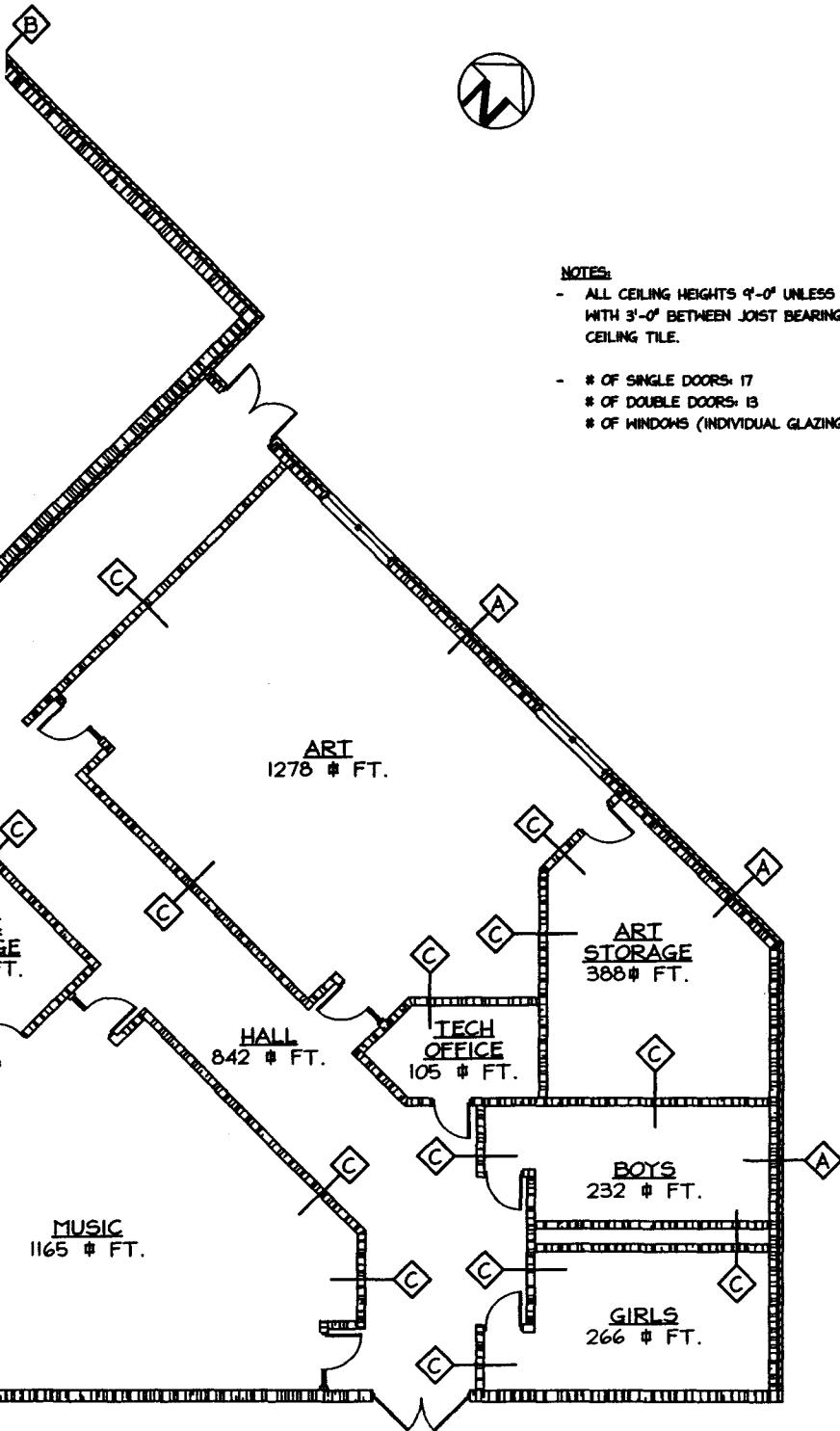
Access to the Carrier gas heat section is particularly difficult. A separation panel inside the unit must be removed and the technician must go into the unit to service the burners.

LENNOX® SOLUTIONS YOU CAN TRUST.®



WALL TYPES:

- (A) 1'-2" WALL: 3 $\frac{1}{2}$ " FACE BRICK, 2" INSULATION, $\frac{5}{8}$ " GYPSUM BOARD SHEATHING, 7 $\frac{1}{2}$ " CONCRETE MASONRY UNITS
- (B) 1'-6" WALL: 3 $\frac{1}{2}$ " FACE BRICK, 2" RIGID INSULATION, $\frac{5}{8}$ " GYPSUM BOARD SHEATHING, 11 $\frac{1}{2}$ " CONCRETE MASONRY UNITS
- (C) 11 $\frac{1}{2}$ " CONCRETE MASONRY UNITS
- (D) 7 $\frac{1}{2}$ " CONCRETE MASONRY UNITS

NOTES:

- ALL CEILING HEIGHTS 9'-0" UNLESS OTHERWISE NOTED, WITH 3'-0" BETWEEN JOIST BEARING & SUSPENDED CEILING TILE.
- # OF SINGLE DOORS: 17
- # OF DOUBLE DOORS: 13
- # OF WINDOWS (INDIVIDUAL GLAZING): 11

FLOOR PLAN
1' = 1'-0"

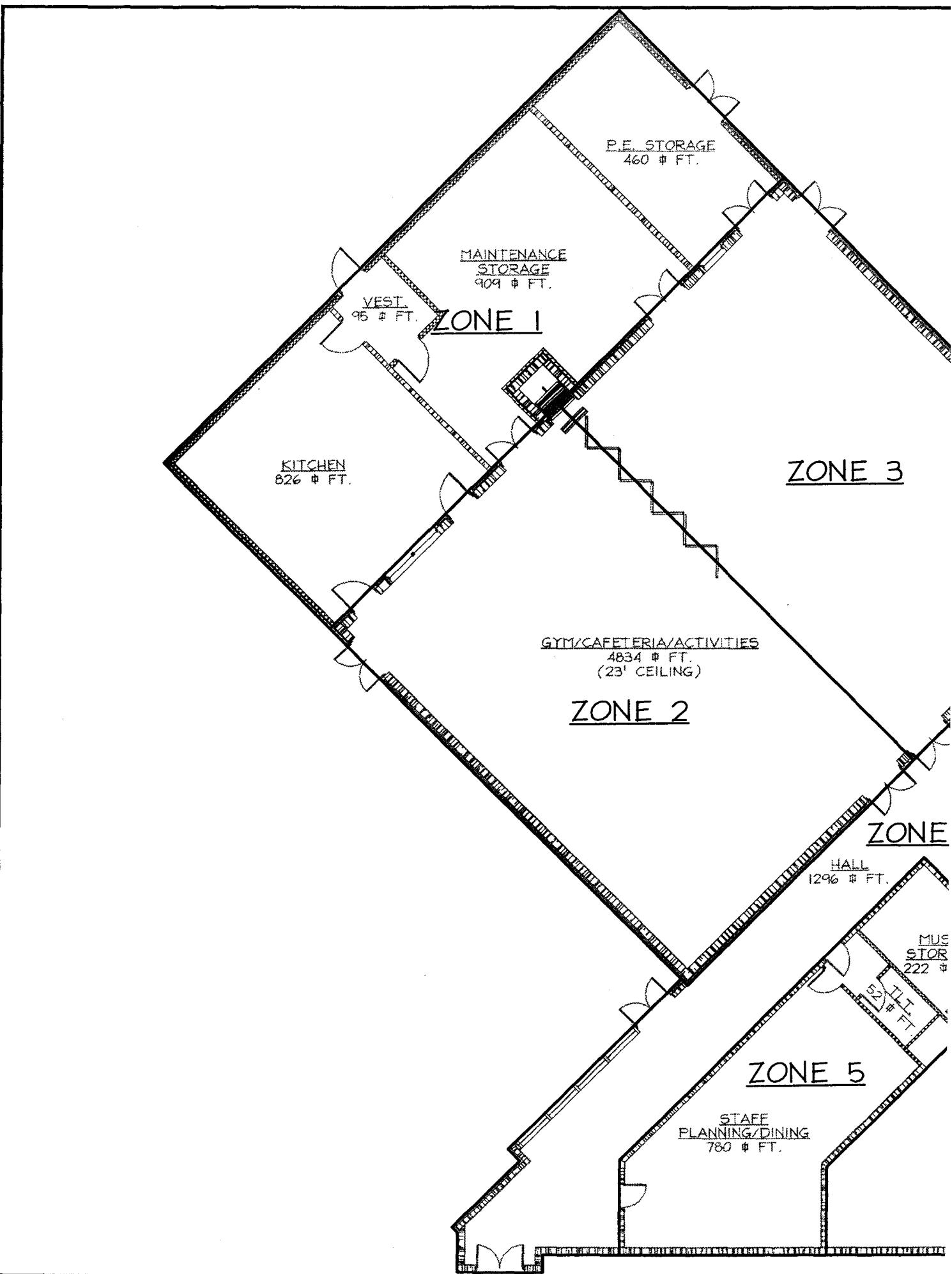
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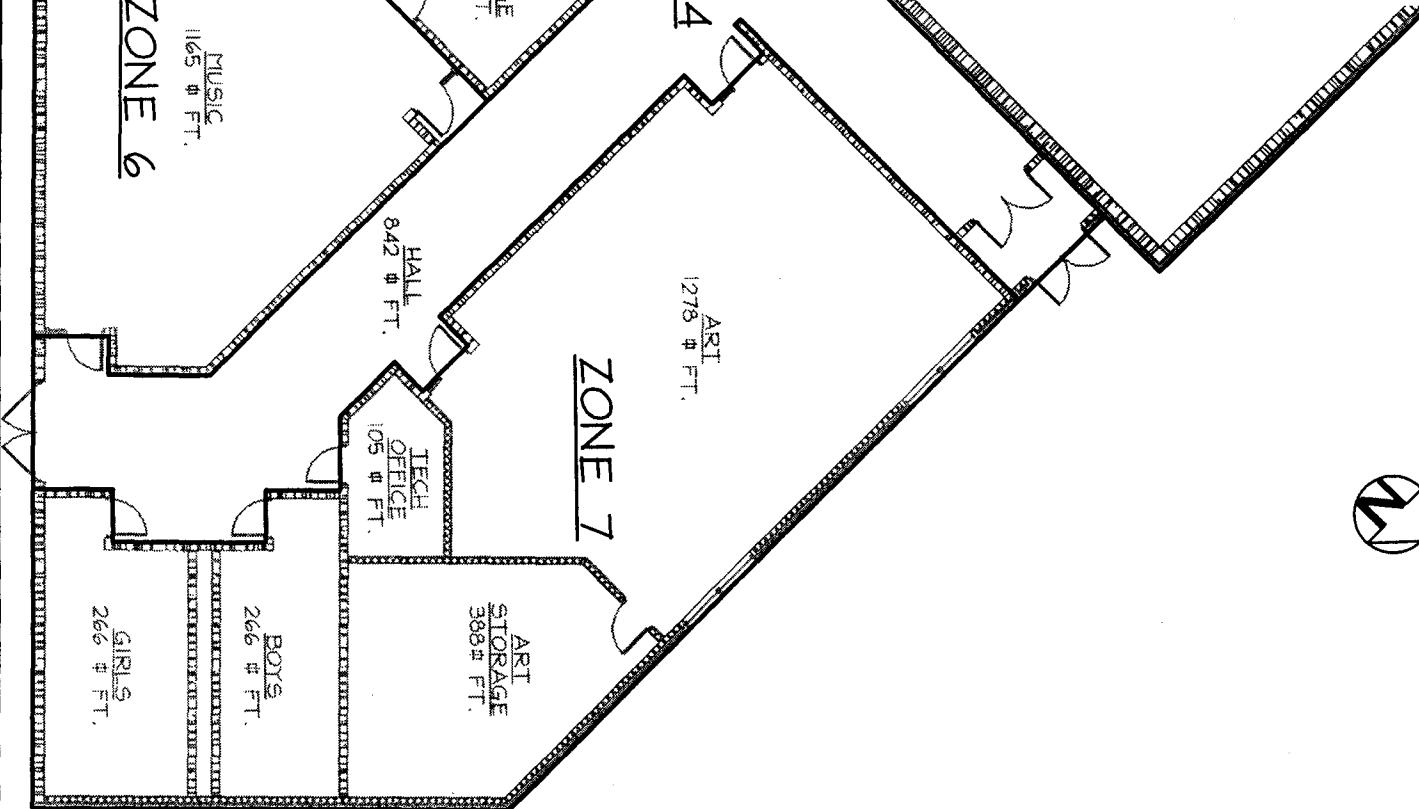
USD 305
SUNSET ELEMENTARY SCHOOL
ADDITION
FLOOR PLAN
MECHANICAL HVAC

DESIGN
SEAN MILLER
KOSUKE ISHIKAWA
SHAWN BEAL
DATE
DECEMBER 10, 2001
SUBMITTAL
FINAL REPORT

SHEET NO.
FIG. 1

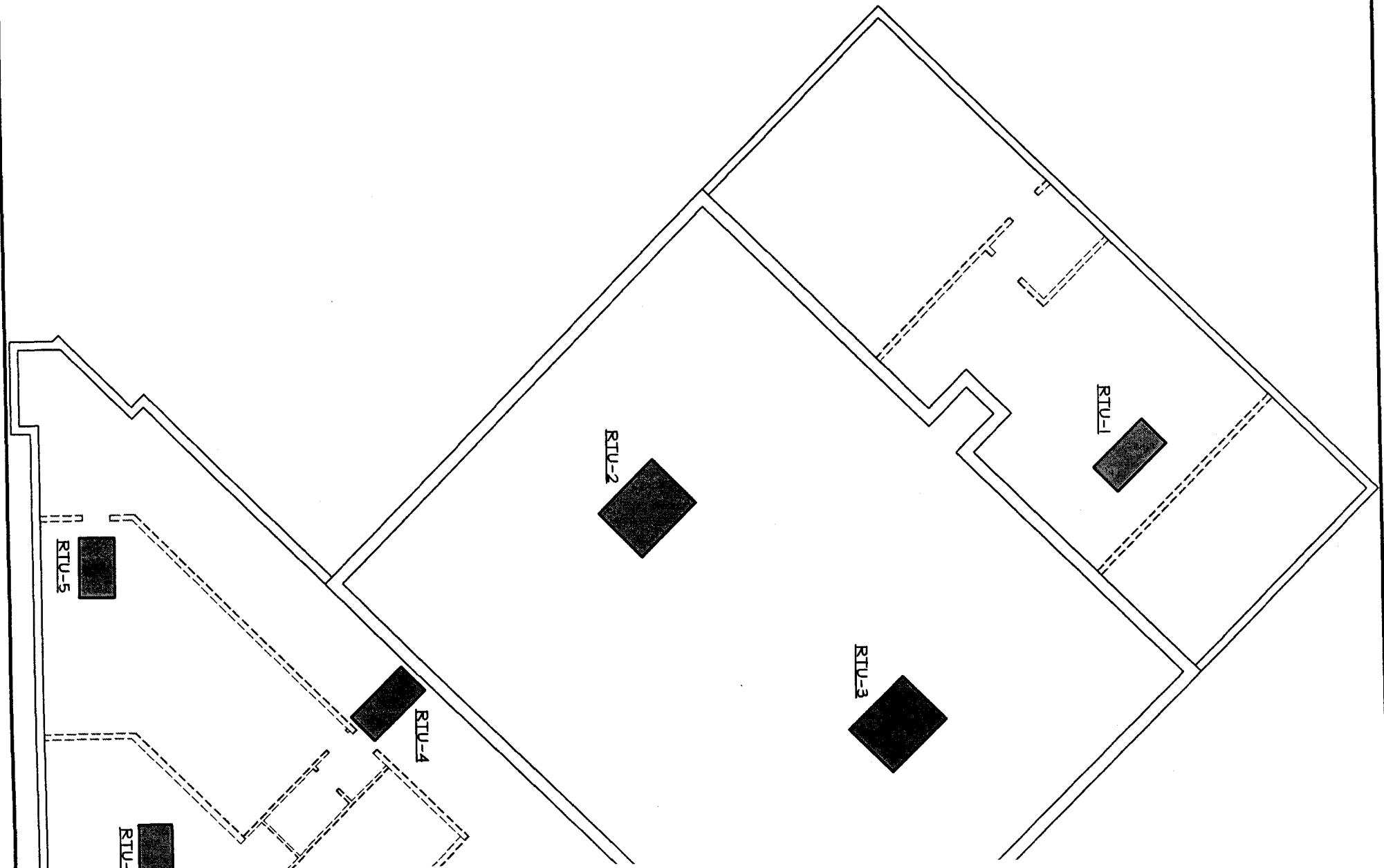


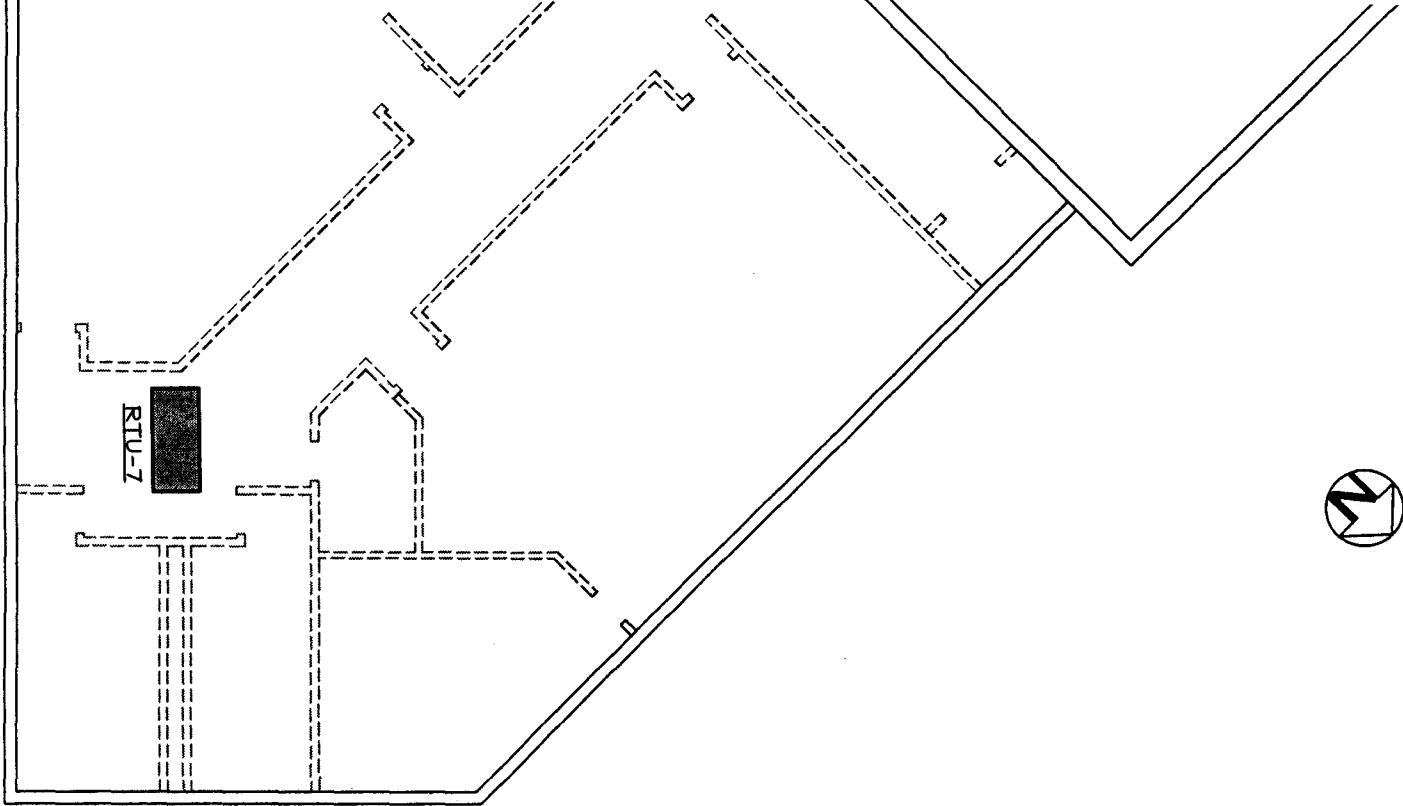




ZONE PLAN

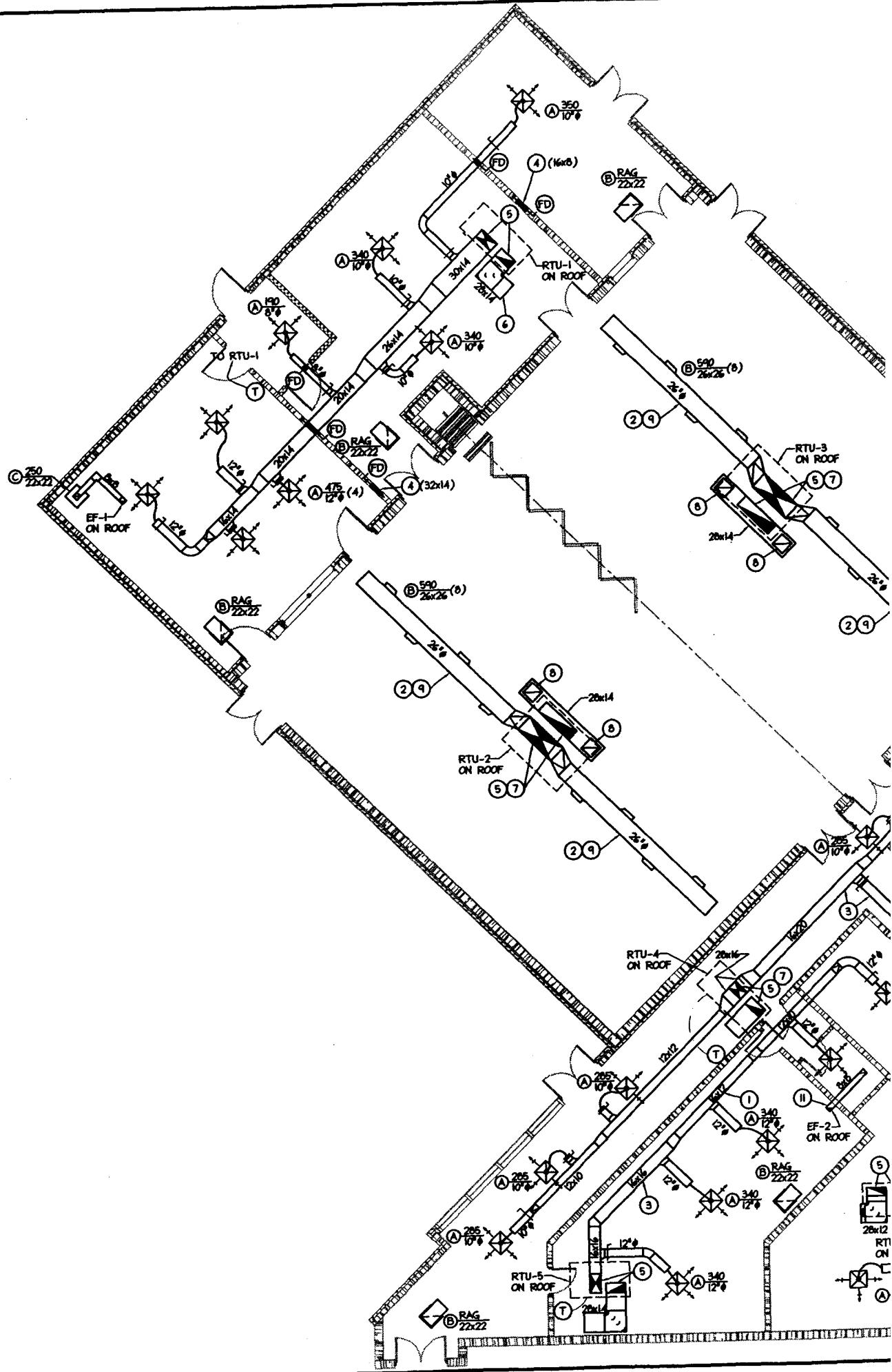
SHEET NO. FIG. 2	DESIGN SEAN MILLER KOSUKE ISHIKAWA SHAWN BEAL	SUNSET ELEMENTARY SCHOOL ADDITION ZONE PLAN MECHANICAL HVAC	ME 750I ADVANCED HVAC DESIGN				NO. _____ REVISION _____ DATE _____ BY _____
DATE DECEMBER 10, 2001	SUBMITTAL FINAL REPORT						

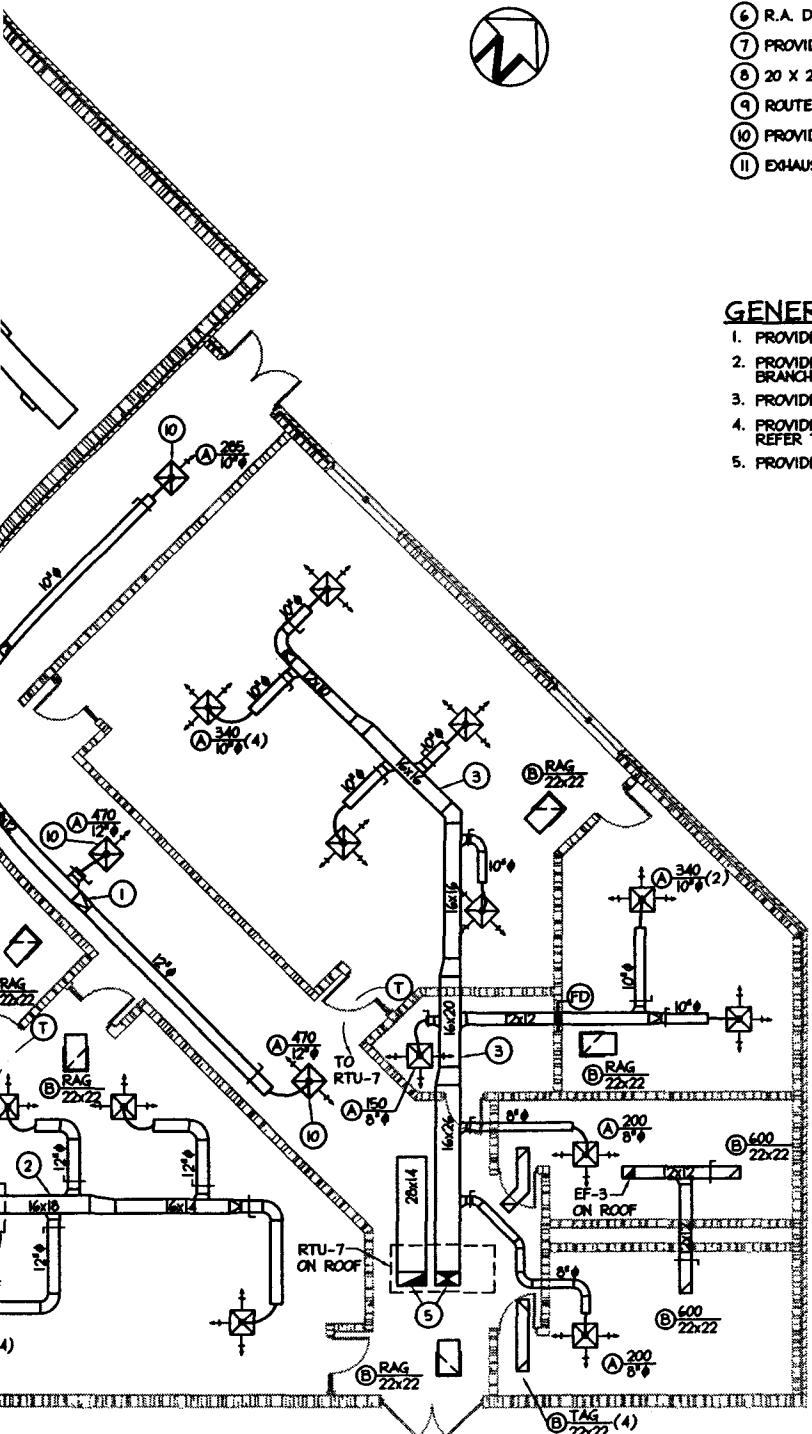




ROOF TOP UNIT PLAN

SHEET NO. FIG. 3	DESIGN SEAN MILLER KOSUKE ISHIKAWA SHAWN BEAL	DATE OCTOBER 26, 2001	SUBMITTER INTERIM REPORT	USD 305 SUNSET ELEMENTARY SCHOOL ADDITION ROOF TOP UNIT PLAN MECHANICAL HVAC	ME 750I ADVANCED HVAC DESIGN		NO.	REVISION	DATE	BY
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PLAN NOTES

- ① SQUARE TO ROUND TRANSITION. (TYP)
- ② ROUTE DUCT IN JOIST SPACE.
- ③ ROUTE DUCT TIGHT TO BOTTOM OF JOIST.
- ④ TRANSFER AIR OPENING IN WALL ABOVE.
- ⑤ S.A. AND R.A. DUCTS UP TO UNIT. TRANSITION AS REQUIRED.
- ⑥ R.A. DUCT OPEN ABOVE CEILING.
- ⑦ PROVIDE 50/50 SPLITTING TEE WITH TURNING VANES.
- ⑧ 20 X 20 OPENING IN TOP OF R.A. DUCT.
- ⑨ ROUTE DUCT CENTERED BETWEEN JOISTS.
- ⑩ PROVIDE SECTORIZING BAFFLES IN DIFFUSER TO PROVIDE THROW INDICATED.
- ⑪ EXHAUST DUCT UP TO FAN ON ROOF, TRANSITION AS REQUIRED.

GENERAL NOTES

1. PROVIDE TURNING VANES IN ALL RECTANGULAR ELBOWS AND OFFSETS, (TYP).
2. PROVIDE 45° HI-EFF. TAKEOFF FITTINGS WITH MANUAL VOLUME DAMPER ON ALL BRANCH TAKEOFFS TO DIFFUSERS. REFER TO DETAIL, FIGURE 6.
3. PROVIDE FLEXIBLE CONNECTION AT ALL MECHANICAL UNITS.
4. PROVIDE INSULATED SOUND BOOTS ON ALL RETURN AIR GRILLES. REFER TO DETAIL, FIGURE 6.
5. PROVIDE ACCESS DOORS FOR ALL FIRE DAMPERS.

DUCT LAYOUT PLAN
1' = 1'-0"

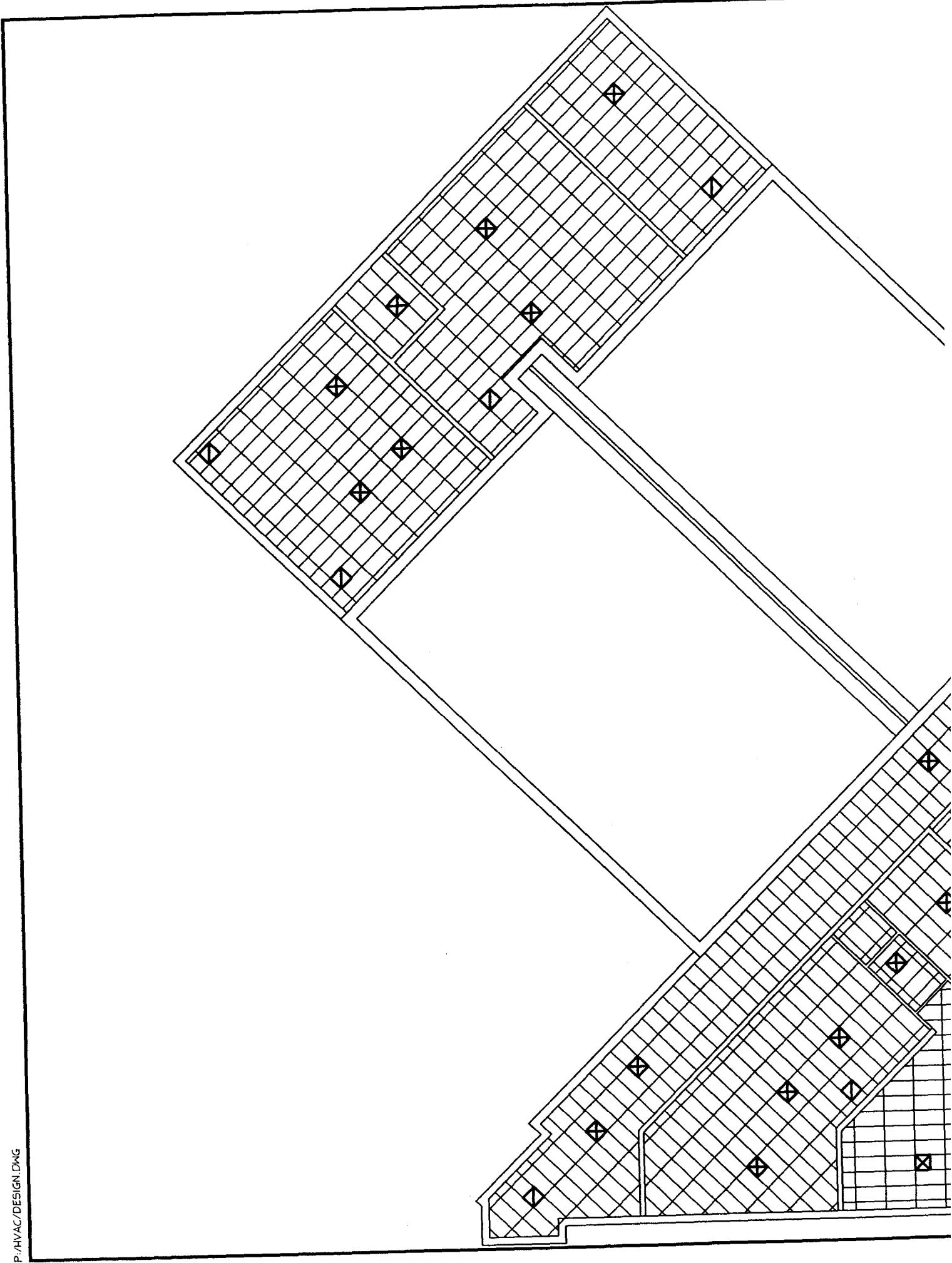
USD 305
SUNSET ELEMENTARY SCHOOL
ADDITION
SCHEMATIC DUCT LAYOUT PLAN
MECHANICAL HVAC

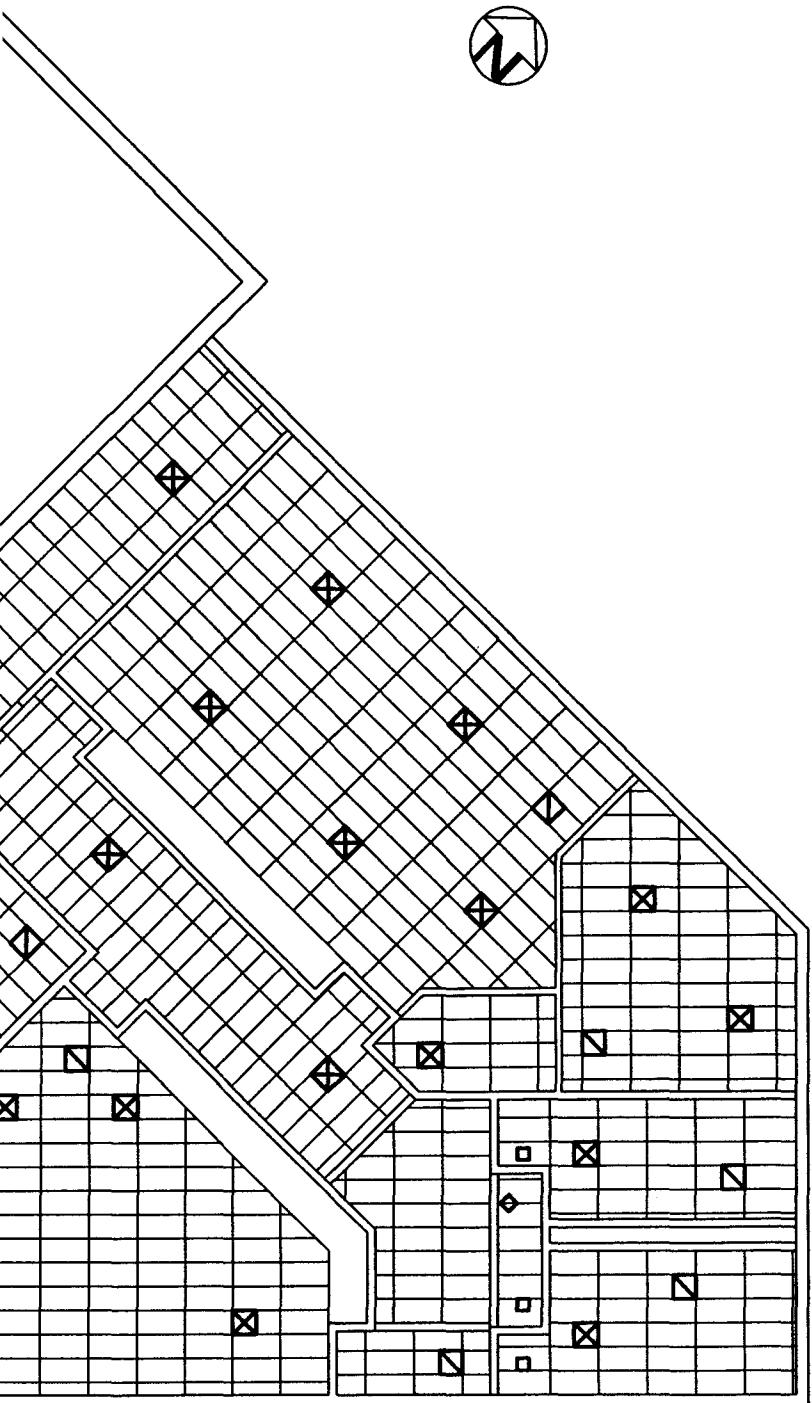
DESIGN	SEAN MILLER KOSUKE ISHIKAWA SHAWN BEAL
DATE	DECEMBER 10, 2001
SUBMITTAL	FINAL REPORT
SHEET NO. FIG. 4	



ME 7501
ADVANCED
HVAC DESIGN

REVISION	
NO.	
DATE	
BY	





PLAN NOTES

- X SUPPLY AIR DIFFUSERS
- RETURN AIR GRILLES
- TRANSFER AIR GRILLES

DIFFUSER PLAN
1' = 1'-0"

USD 305
SUNSET ELEMENTARY SCHOOL
ADDITION
DIFFUSER LAYOUT PLAN
MECHANICAL HVAC

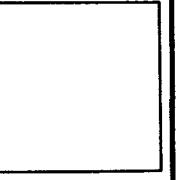
DESIGN
SEAN MILLER
KOSUKE ISHIKAWA
SHAWN BEAL

DATE
DECEMBER 10, 2001

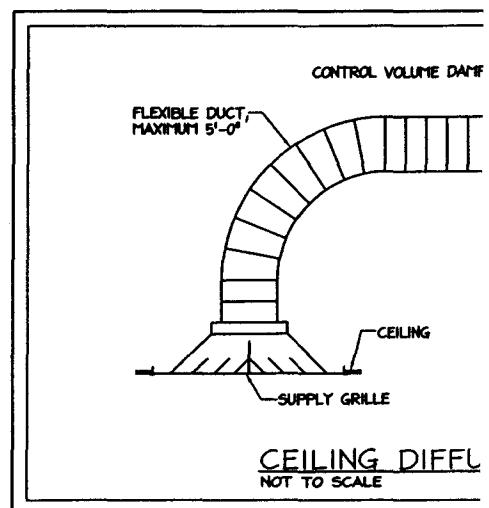
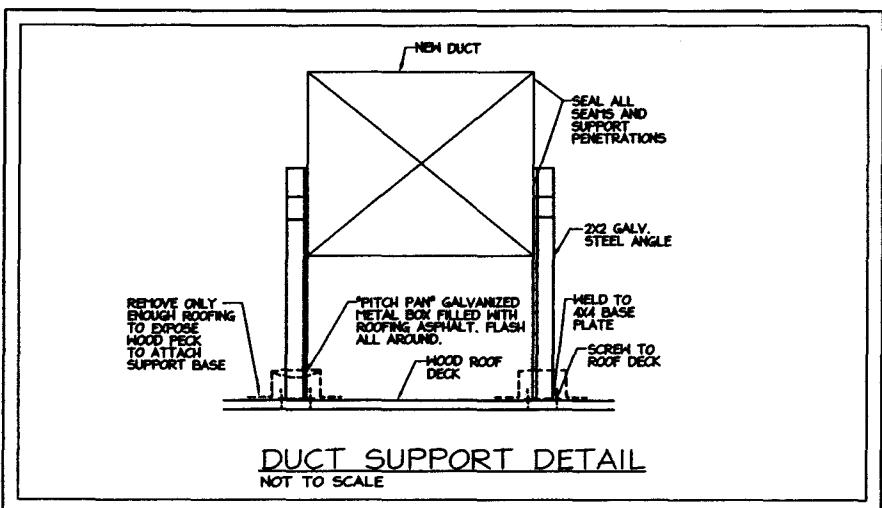
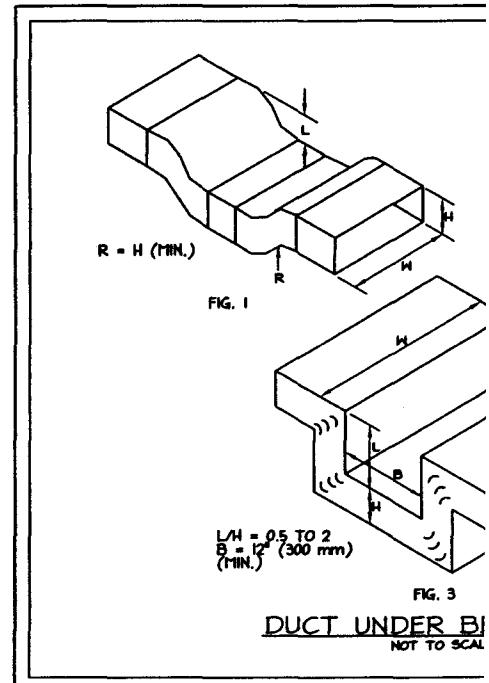
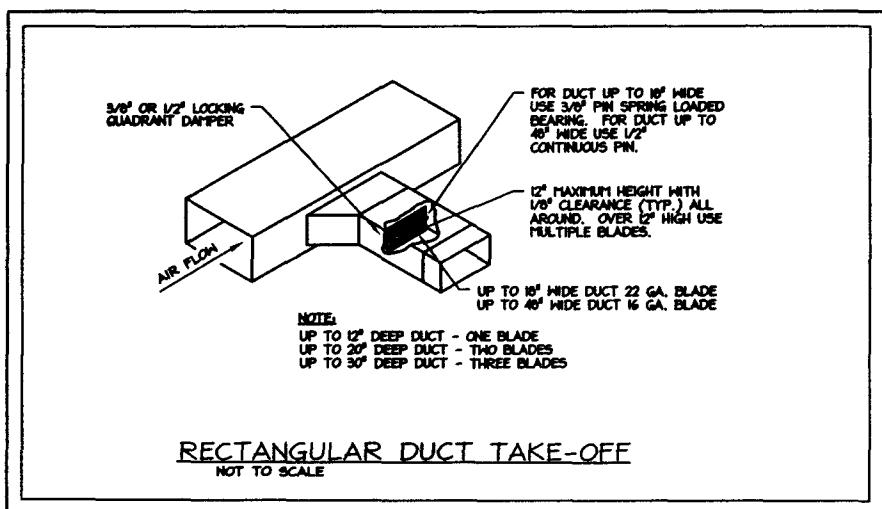
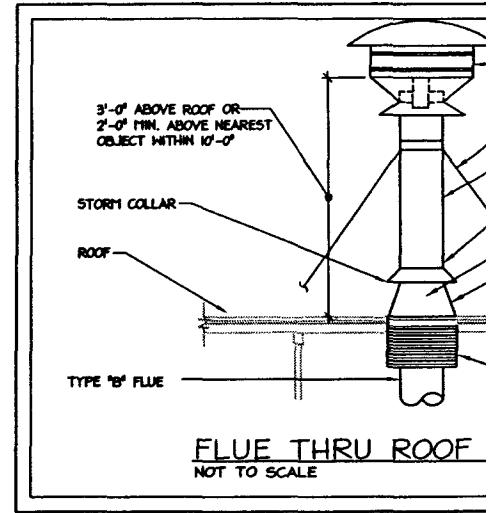
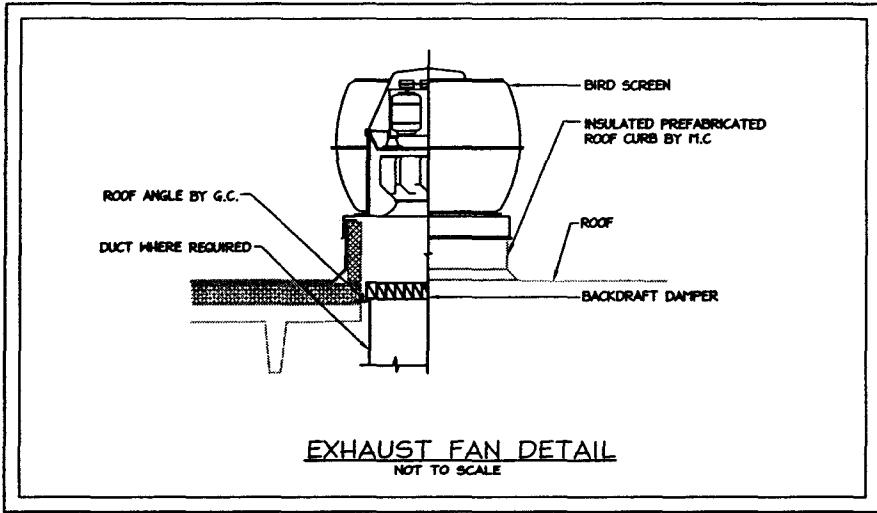
SUBMITTAL
FINAL REPORT

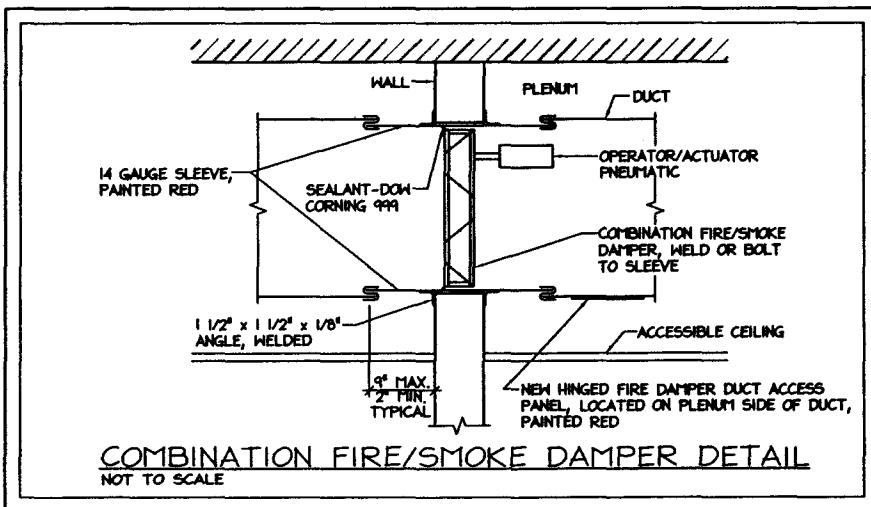
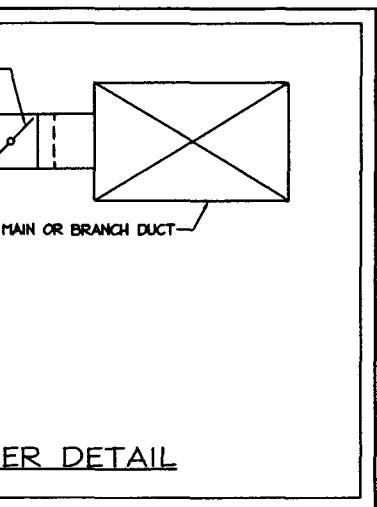
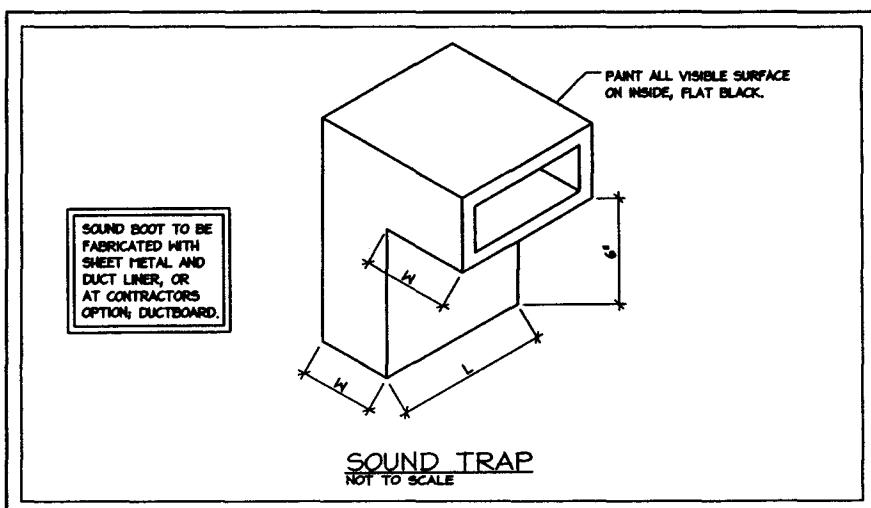
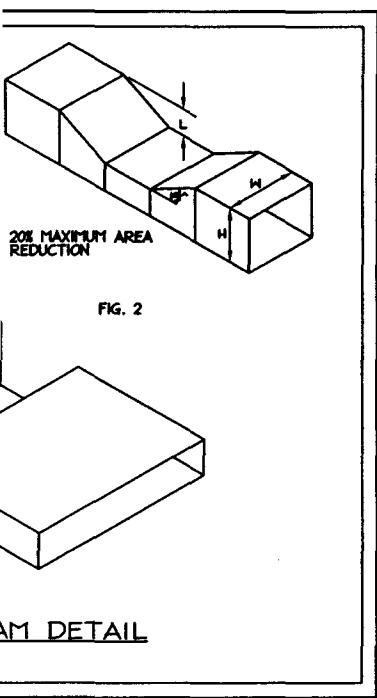
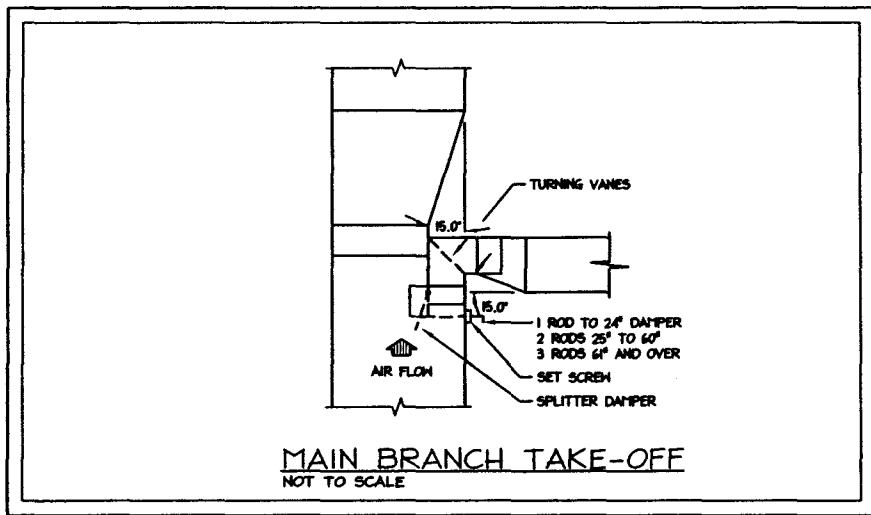
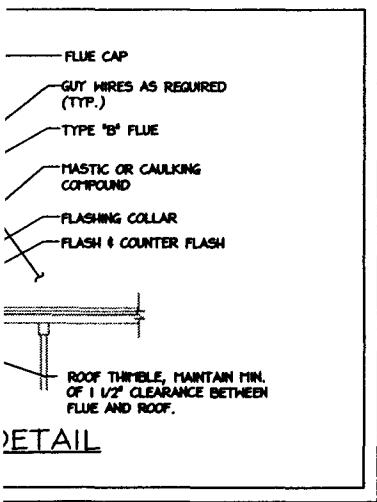
FIG. 5
SHEET NO.

ME 7501
ADVANCED
HVAC DESIGN

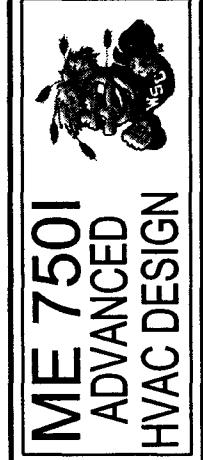


REVISION NO.
DATE BY





REVISION	DATE	BY



USD 305
 SUNSET ELEMENTARY SCHOOL
 ADDITION
 MISCELLANEOUS DETAILS
 MECHANICAL HVAC

DESIGN	SEAN MILLER KOSUKE ISHIKAWA SHAWN BEAL
DATE	OCTOBER 26, 2001
SUBMITTAL	INTERIM REPORT
SHEET NO.	FIG. 6